Impact Response and Damage Characteristics of Carbon Fiber Reinforced Aluminum Laminates Under Low Velocity Impact Loading

Khalid M. Al gadah, Abdulaziz S. Alaboodi

Abstract: CARALL hybrid material has been extensively used in the aircraft structure due to their competitive impact strength. Low velocity impact test is utilized to evaluate the impact and damage properties for such material. It is also employed to observe complex damage mechanisms. A numerical modelling is an alternative way for impact assessment. This paper investigates the impact and damage properties under low velocity impact using numerical modeling and experimental work. A three-dimensional (3D) finite element (FE) model was developed and validated with two studies from the literature. This model was meshed with solid elements. It was subjected to 2.4 m/s impact velocity and to 10 J impact energy. Absorbed energy, penetration, impact load and damage morphology were obtained. The impact energy was efficiently absorbed by the material. Both aluminum alloy layers underwent plastic deformation whereas the fiber layer failed. A macroscopic cross-sectional morphology was presented using the FE model. An agreement between the numerical and the experiment results were achieved and discussed.

Keywords: Impact properties; Damage characteristics; Carbon fiber reinforced aluminum laminate (CARALL); Fiber metal laminates (FMLs); Low velocity impact (LVI).

I. INTRODUCTION

As a result of the improvement and development in the aircraft industry in terms of performance and weight saving, the fiber metal laminates (FMLs) progress was widely noticed. FMLs is a novel material used in aircraft applications due to their unique mechanical properties under impact, flexure and tensile events [1]. Delft University has developed FMLs to withstand fatigue loading. The most common used FMLs are carbon fibers reinforced aluminum laminate (CARALL) which is based on carbon fiber, glass fiber reinforced aluminum laminate (GLARE) based on fiber glass, and aramid fiber reinforced aluminum laminate (ARALL) based on aramid fibers. CARALL is constructed from multiple carbon fiber plies bonded together with aluminum plies. Aircraft wing and fuselage are typically made form GLARE. The upper fuselage sheet is built from GLARE which consequently save 794 kg of the Airbus A380 weight [2]. FMLs afford substantial impact resistance, high fracture toughness, energy absorption capability, moisture resistance, weight savings, low density, corrosion resistance, fire resistance, and less repair and less maintenance cost and time [3].

There are different types of failure modes occur in CARALL material. For example, interfacial debonding and inter-laminar delamination, fiber breakage, matrix cracking, metal cracking and fiber pull out are all damage forms. Fiber bridging effect arrest the crack instigation and propagation in FMLs [4]. A compressive force on FMLs causes delamination buckling. Delamination zone propagation and layer buckling diminishes the residual strength [5]. Specific penetration energy of CARALL composite laminates declines when the thickness increases [6]. In [7] GLARE 5-2/1-0.3 was fabricated and tested with repetitive impact loading under low velocity regime. This FMLs shows outstanding impact resistance attributed to the aluminum layers which arrest the delamination evolution. Pervasive delamination and several matrix cracking occur in carbon fiber reinforced epoxy subjected to low velocity impact loading [8]. GALRE impact resistance was compared with aluminum and carbon / polyetherimide (PEI) laminates [9]. GLARE provides optimal properties for instance, significant residual strength, impact, fatigue and corrosion resistance. The internal degradation and plastic defamtion which occur during FMLs impact damage is very complex mechanisms [10]. Moreover, Kevlar layers, for example, when it hybridize with FMLs fiber its impact strength and absorbed energy increase. This happens when the impact force subjected toward the Kevlar layers side [11]. It was found that zero alignment of the Kevlar fibers and glass fibers could offer sustainable outcomes in terms of modulus of elasticity value under tensile strength [12]. When the fiber plies are oriented in the loading direction, they provide high tensile strength, yield strength and modulus of elasticity [13]. Many AL2024 layers were utilized for GLARE laminates allowed for superlative residual and blunt notch strength as well as impact and fatigue resistance [14].

Finite element analysis (FEA) can simulate the impact behavior of CARALL composite laminates. Macroscopic and mesoscopic are two approaches widely used for composite materials simulation subjected to impact loading [15,16]. With regard to failure criteria the element stiffness declines once it fails. Convergence and instability issues arise during simulation as a result of element distortion caused by stiffness reduction [17].

Complex damage mechanisms are observed using numerical simulation.
This paper presents an advanced 3D finite element model used for evaluating of CARALL impact and damage properties under low velocity impact. This model was tested and validated with two studies from the literature. They were named as CARALL 1 as for the numerical model of this study, CARALL 2 for [18] experiment and CARALL 3 for [19] work. It was meshed with connivant nods and elements until it provided meaningful results. It was exposed to 2.4 m/s impact velocity and to 10 J impact energy. The outcomes properties were: absorbed energy, maximum penetration, maximum impact load, fiber brakeage and delamination. The numerical model was validated with related studies.

II. EXPERIMENT SETUP

The test procedure and material fabrication method of the specimen were adopted form Rajan et al. experimentatiation [18]. It was considered to be consistent with its geometry and impact parameters values in order to get the numerical model validated thereafter.

A. Material fabrication

The sample was made from unidirectional carbon fiber fabric 300 gsm weight blended with 0.3 mm two aluminum sheets. For more effective joining condition between layers, ASTM D 2674 and ASTM D 2651 standards were employed. The carbon fiber ply was oriented in 0° direction. The sample was manufactured using hand layup method. The carbon fiber layer was impregnated into the aluminum layers. An Epotec YD mixed with hardener triethylenetetramine (TETA) was used as a resin. It was wiped between the metal and fiber layers. The sample was cut and machined in a rectangular shape according to 150 mm x 100 mm ASTM standards. Figure 1 illustrates the schematic drawing of 2/1 CARALL.

![Schematic diagram of 2/1 CARALL](image)

Fig. 1. Schematic diagram of 2/1 CARALL

B. Impact test

Low velocity impact test was performed on a CARALL 2/1 sample. Figure 2 shows the drop tower apparatus used for impact test. ASTM D 7136 standard was followed. Four samples were tested in [18] to obtain more accurate results. The impactor velocity was specified 2.4 m/s which correspond to 10 J. The impactor was left up to 500 mm. 2.5 kg loads were added onto the machine cross head.

![Impact test setup](image)

Fig. 2. (a) Drop tower machine, and (b) its schematic drawing [18].

III. NUMERICAL SIMULATION

The test procedure and material fabrication method of the

A. Finite element analysis

3D model was developed and simulated using finite element analysis. There are different types of nodes and elements generally used in finite element analysis. They are illustrated in figure 3. For bar and truss components one-dimensional nodes and element are used. Two-dimensional nodes and element are utilized for 2D problem such as plain stress and plain strain. Thus, for CARALL model 3D hexahedral elements were dominantly used.

![Nodes and elements types in FEA](image)

Fig. 3. Nodes and elements types in FEA [21]

There are two approaches based on geometry size can be followed for model simulation. They are called macroscopic and mesoscopic simulation. Macroscopic scale does not observe more details such as, wrap and weft yarns interaction. They are considered as one object having orthotropic mechanical properties. In this simulation, shell elements are employed. This is consequently reducing the simulation time. On the other hand, mesoscopic approach can simulate more detail mechanisms, but it consumes more computational cost and time [15,16]. Hashin’s failure criteria and elastic-plastic property would simulate the aluminum layers of the fiber metal laminates under the low velocity impact successfully [20]. Composite layer under an impact loading is computed using the following equations [22]:

\[
[P] = [K][d] + [C][\dot{d}] + [M][\ddot{d}]
\]

(1)

\[
[K] = \epsilon [A][B]^T[D][B][A]
\]

(2)

Where, \([P]\) is the impact force, \([K]\) is the stiffness matrix, \([d]\) is the displacement matrix, \([\dot{d}]\) is the velocity matrix, \([\ddot{d}]\) is the acceleration matrix, \([M]\) is the mass matrix, \([C]\) is the damping matrix, \([B]\) is geometric properties matrix , \([D]\) is the material property matrix, \(A\) is the layer area and \(\epsilon\) is layer thickness.

In addition, material property matrices of isotropic and orthotropic materials are assigned according to Hooke’s law as followed [22,23]:

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Where $\nu$ is the Poisson’s ratio and $E$ is the Young's modulus. $\sigma$ is the normal stress and $\tau$ is the shear stress. $G$ represents the shear modulus, $F$ is the shear strain and tensile strain is designated as $\varepsilon$. Next, the numbers 1 and 2 indicate the direction in $i$ and $j$ respectively.

### B. Geometry and materials

Specimen geometry was design similar to Rajan et al. experiment [18]. Three-dimensional model was created and meshed with 6913 nodes and 4738 elements. It was design with size equals 150 mm x 100 mm. The sample was constructed from two aluminum layers and one carbon fiber/epoxy layer. Aluminum layer thickness is 0.3 mm. The specimen total thickness is 1.2 mm. The impactor was drawn with a hemispherical tip had 12.7 mm diameter. Large impactor diameter causes short contact time, big damage and high applied stresses [29]. The materials properties were taken from the Ansys software materials library. AL2024T351 was used for metal layer. Table 1 demonstrates the material properties and Johnson Cook parameters of AL2024-T351. Moreover, unidirectional wet carbon fiber/epoxy (230 GPa) was used for the fiber layer. The carbon fiber properties are shown in table 2. High speed tool steel material was assigned to the impactor. Its mechanical properties are illustrated in table 3. Instead of 8.16 g density of the high-speed tool steel, the density was calibrated. It was maximized until the impactor mass reached 3.5 kg. The reason behind that is to match the impact velocity and the impact energy values in the experiment. The other way to achieve similar impact energy and velocity is to draw more components. However, there is no necessity to draw extra loads on the model where the simplicity is a plus in modelling and simulation.

### Table- I: Material properties and Johnson Cook parameters of AL2024-T351

<table>
<thead>
<tr>
<th>Property</th>
<th>Value</th>
</tr>
</thead>
<tbody>
<tr>
<td>Density (g/m$^3$)</td>
<td>2.785</td>
</tr>
<tr>
<td>Initial yield stress (MPa)</td>
<td>265</td>
</tr>
<tr>
<td>Hardening constant (MPa)</td>
<td>426</td>
</tr>
<tr>
<td>Hardening exponent</td>
<td>0.34</td>
</tr>
<tr>
<td>Strain rate constant</td>
<td>0.02</td>
</tr>
<tr>
<td>Thermal softening exponent</td>
<td>1</td>
</tr>
<tr>
<td>Melting temperature C</td>
<td>501.85</td>
</tr>
<tr>
<td>Reference Strain Rate(1/sec)</td>
<td>1</td>
</tr>
<tr>
<td>Shear modulus (MPa)</td>
<td>27600</td>
</tr>
</tbody>
</table>

### Table- II: Material properties UD carbon/epoxy (230 GPa) Wet

<table>
<thead>
<tr>
<th>Property</th>
<th>Value</th>
</tr>
</thead>
<tbody>
<tr>
<td>Density (g/m$^3$)</td>
<td>1518</td>
</tr>
</tbody>
</table>

### Table- III: Material properties of tool steel high speed

<table>
<thead>
<tr>
<th>Property</th>
<th>Value</th>
</tr>
</thead>
<tbody>
<tr>
<td>Density (g/m$^3$)</td>
<td>853*</td>
</tr>
<tr>
<td>Tensile yield strength (MPa)</td>
<td>2170 MPa</td>
</tr>
<tr>
<td>Tensile ultimate strength (MPa)</td>
<td>2390 MPa</td>
</tr>
</tbody>
</table>

*Modified value.

Moreover, Lagrangian finite element algorithm was chosen to analyses the impact event. The impactor was aligned close to the top surface of the first aluminum layer. This is to reduce the simulation time. The friction coefficient between the impactor and CARALL 1 top layer could be set 0.2 for design validation purpose [24]. However, the contact mechanism is beyond the area of interest in this research. Hence, it was set as frictionless between all the objects. The CARALL layers and the impactor were allowed to interact. The mesh was refined until no energy error occurred. Sweep meshing method was utilized for meshing. For source or target choice, automatic thin was selected. 3D solid element was chosen. Quad or tri was assigned for free face mesh type. The model was analyzed under 22 C°.

### C. Meshing

Figure 4 shows the 3D drop test model after generating mesh. Solid hexahedral elements were assigned to the four objects namely the impactor, the aluminum top ply, the carbon fiber reinforce polymer (CFRP) ply and the aluminum bottom ply. The sweep mesh was implemented and located all around the impactor region.

![Fig. 4. Impact test 3D meshed model](image-url)
D. Initial conditions

To obtain accurate results, the boundary conditions were matched with the actual experiment.

The initial velocity was 2.4 m/s. The end time was set 6.5 ms. Maximum number of cycles was given 1E+07. Beam element solution type was chosen bending. The impactor was considered having elastic behavior. It can be chosen as a rigid body. However, after setting its behavior as a rigid body, energy error was kept occurring. Hence, it was given an elastic behavior to simulate the real-life event. The initial velocity of the impactor could be figured out utilizing the following equation [22]:

\[ V_i = \sqrt{2g(h - d)} \]  

Form the equation 5, \( g \) is the gravitational acceleration, \( h \) is the impactor height, \( d \) is the displacement between the impactor insert tip and the test sample.

E. Meshing

The absorbed energy by the specimen is calculated using equation number 6:

\[ E_{abs} = E_{imp} - E_{res} \]  

Where,

\[ E_{imp} = \frac{1}{2} m V_i^2 \]  

From equation 6 and 7 \( E_{abs} \) is the absorbed energy, \( E_{imp} \) is the impact energy, \( E_{res} \) is the residual energy, \( m \) is the impactor mass and \( V_i \) is the initial velocity. The applied force could be obtained by triggering the external forces or differentiating impact energy/ kinetic energy – displacement curve. The absorbed energy, deformation, applied force were considered outcomes in this numerical model.

IV. RESULT AND DISCUSSION

The result of three carbon fiber reinforced aluminum laminates are compared and discussed. The CARALL 1 represents the numerical model. CARALL 2 is the sample utilized in [18]. CARALL 3 is adopted from Yao et al. research [19]. The impact properties and damage characteristic of this material is presented in this part.

A. Impact properties

The sample penetration, maximum impact load and absorbed energy are the studied properties. Table 4 demonstrates the impact properties of CARALL 1 and CARALL 2. The maximum load of each is different. High impact load 3390 N is applied in the numerical analysis. In the opposite, the maximum load on the CARALL 2 was almost one-fourth of CARALL 1. It can be a consequence of different aluminum layer properties. The aluminum properties of CARALL 2 is not mentioned in [18]. Nonetheless, there is an agreement found with the result in Yao et al. work [19]. This is demonstrated in figure 9 in the absorbed energy subsection.

The maxim penetration at the maximum load is 7.7 mm. This displacement is not far away from 5.9 mm. There is quite agreement reached with all properties except the absorbed energy which could be linked to the fabrication procedures.

B. Penetration

Figure 5 shows the maximum deformation of CARALL 1 under 10 J impact energy. The more deformed area is the one that around the impactor tip. The maximum deformation is 7.7 mm as it appears in the legend. It happens at 4.2 ms of the impact event.

C. Maximum load

Kinetic energy verses displacement curve was plotted to find out the impact load. Figure 6 shows the kinetic energy-displacement plot. From the graph underneath the kinetic energy with the value of 10 J is set as an initial impact energy.

It decreases with the change in displacement. As the impact energy increases the laminates displacement, damage area and contact force increase [30]. At the end of the impactor travel, the kinetic energy reached zero. After that, the impactor rebound takes place. This causes both drop in displacement and raise in kinetic energy. The decrease in displacement occurs due to the plastic reaction by the CARALL 1 while the impactor is moving upward. Beyond the plastic deformation limit, the material will not return to its original shape even if the applied load is completely removed. The energy that is consumed due to plastic work is 4.632 J. Therefore, maximum load is the derivative of the kinetic energy with respect to displacement.

<table>
<thead>
<tr>
<th>Property</th>
<th>Numerical/ CARALL 1</th>
<th>Experimental/ CARALL 2 [18]</th>
</tr>
</thead>
<tbody>
<tr>
<td>Specimen thickness (mm)</td>
<td>1.2</td>
<td>1.2</td>
</tr>
<tr>
<td>Impact energy (J)</td>
<td>10</td>
<td>10</td>
</tr>
<tr>
<td>Impact velocity (m/s)</td>
<td>2.4</td>
<td>2.4</td>
</tr>
<tr>
<td>Penetration at maximum load (mm)</td>
<td>7.7</td>
<td>5.9</td>
</tr>
<tr>
<td>Maximum load (N)</td>
<td>3390</td>
<td>730</td>
</tr>
<tr>
<td>Absorbed energy at maximum load (J)</td>
<td>10</td>
<td>6.1</td>
</tr>
</tbody>
</table>
The load verses displacement curve is plotted and demonstrated in figure 7. The peak values CARALL 1 and CARALL 3 deviate from CARALL 2. The trend of CARALL 1 show fluctuation varies with the time. Oscillation of the load-time magnitude indicates failure symptoms in the material structure. The increase in force-time magnitude means an increase in bending stiffness [10]. Bottom peaks imply materials failure while exposing to impact damage. Most interestingly, before 6 ms all the sample show increase in the magnitude. This might be because of the resistance of the aluminum bottom layer. The end time of is each experiment is different. CARALL 1 and CARALL 3 withstand high impact load.

The absorbed energy can be computed from the area under the load-displacement curve. As it is presented in figure 8, the CARALL 1 impact load begins with an enormous ductile behavior. Next, it shows somewhat linear increase corresponding with displacement due to elastic deformation up to 6 mm roughly. There is a sudden drop in the load can be related to debonding effect between the fiber and aluminum layers. Before reaching the peak load there is a sharp rise which can be interpreted as a brittle behavior of the CFRP layer. Fiber breakage occurs as a result of the reduction in bending stiffness [18]. At 7.3 mm perforation takes place on the CFRP layer. It happens at 4.9 ms and 825 N load which is beyond the maximum load. In the meantime, the impactor returns to the opposite direction. The load and displacement start to decrease. The change in load with respect to the time is fairly associated with the absorbed capacity of the materials. Therefore, integration of the area under the curve is equal to the absorbed energy by the material.

The absorbed energy while impacting the CARALL 1 with the impact load of 3390 N is 10J. The absorption efficiency of which equals 100%. In fact, the sample absorbed up to 10.4 J as it depicted in figure 9. This is slightly higher than the impact energy. Nevertheless, this increase in absorbed energy might be attributed to the inflexion that happens when the plastic wave reaches the fixed sides of the specimen [25]. It can be a result of the stored energy inside the material. At the maximum impact energy, some of the energy is stored in the FMLs whereas other return to the system and propagates to form multi different damage modes and plastic defamation on the aluminum layers [26]. There is a similarity between in the peak values, CARALL 2, on the other hand, consumes most of the energy via delamination penetration and perforation [28]. Thus, it is noticed that the absorbed energy and impact time are not dependent on the impact energy [27].
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E. Damage characteristics

The main damage mode observed among the samples is delamination. Under low impact energy most of the energy is dispelled in form of matrix damage, interfacial and inter-laminar delamination. In contrast, matrix cracking, fiber fracture, metal cracks and perforation consume the entire energy under low impact regime [19]. The delamination length is rather similar as it is shown in the cross-section view of figure 11. Carbon fiber layer conveys the energy backward which causes more damage on the back side of the sample [18]. Fiber plain orientation influences the impact resistance of the whole material [20]. Voids which initiates while hand lay-up process influence the mechanical properties [26]. Consequently, the interfacial bonding has a significant influence on the CARALL behavior.

F. CARALL 1 morphology

FE simulation provides exclusive features to observe complex mechanisms. Figure 12 depicts the morphology of the CARALL 1 layers independently. The highest deformation appears on the Al bottom layers. The Al top layer shows the lowest displacement value as it is located at the top position of the specimen. This variance in deformation is caused by the transverse shear throughout the layers thicknesses [26]. There is a perforation in the fiber layer. The general mechanisms occur under impact are aluminum yielding at the impactor tip, large strain causes fiber facture and aluminum cracking related to perforation of the CARALL layers [31]. The failure of the inner layer causes a degradation in impact resistance which cannot be easily seen by the naked eye.

Table 1: Damage properties of CARALL 1 and CARALL 2

<table>
<thead>
<tr>
<th>Property</th>
<th>Numerical/ CARALL 1</th>
<th>Experimental/ CARALL 2 [18]</th>
</tr>
</thead>
<tbody>
<tr>
<td>Perforation</td>
<td>Fiber layer only</td>
<td>Full perforation</td>
</tr>
<tr>
<td>Delamination</td>
<td>Yes</td>
<td>Yes</td>
</tr>
<tr>
<td>Metal crack</td>
<td>No</td>
<td>Yes</td>
</tr>
<tr>
<td>Fiber fracture</td>
<td>Yes</td>
<td>Yes</td>
</tr>
</tbody>
</table>

V. CONCLUSION

A 3D numerical model was developed based on finite element method to simulate carbon fiber reinforced aluminum laminates under low velocity impact. The model was validated with two experiments found in the literature. Impactor velocity was set 2.4 m/s. The impact energy was 10 J. The maximum impact load reached 3390 N. The material defamation was 7.7 mm. Perforation took place in the CFRP layer whereas the aluminum top and bottom layers did not show any metal fracture.
Interfacial delamination and fiber fracture occurred in CARALL 1. In addition, CARALL 2 diverged in terms of the impact load. CARALL 3 had quite similarity in the impact energy and absorbed energy values. There was an increase in the impact energy attributed to the fixture mechanism of the sample. This is recommended to future investigation. The relationship between the impact energy and residual energy can be considered as well. The FE modelling is an effective tool for determining the impact properties and damage characteristics of the CARALL material.

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