

Buckling Behavior of Sandwich Panel with Isotropic Core and Orthotropic Faces



Saheel Arshad, Abhay Kumar Chaubey

Abstract: A sandwich panel is a lightweight structure, economical and having low thermal conductivity. It is made up of three layers in which the middle layer is called core which is bounded with thin layers at top and bottom called faces. Generally, the core has relatively low-density which makes it lightweight. The buckling load analysis of sandwich panel having isotropic core and orthotropic faces is studied for different support conditions using the FE based software ABAQUS. The FE model is validated with suitable published results. Then it is used to find critical buckling load of sandwich panel with isotropic core and orthotropic faces. Many new results have been presented for different thickness, end conditions, aspect ratio, etc.

Keywords: Terms: sandwich panel, buckling behavior, ABAQUS

I. INTRODUCTION

Sandwich panels (Fig. 1 and Fig. 2) subjected to uniaxial buckling are used in numerous engineering applications like structural, marine, aerospace, and others. Sandwich panel is used as a roofing unit in the airport, hanger, large column-free area, and marine structure. Sandwich panels are prominent because of the lightweight and low thermal conductivity. Different boundary combinations of Sandwich roofs/walls are used in construction practice. Buckling of Sandwich panels subjected to compressive loads is one of the normal failure modes and it is important to calculate the crucial buckling load for the safe design of the structure.

Construction of the building is a big challenge to civil engineers and the government especially in developing countries throughout the world. This problem is due to the dense concentration of the population in urban areas. It is a very difficult task to meet the challenges in the construction practices; however, it is a big challenge to the civil engineers to meet the housing demand in a short span without compromising on the quality. Traditional building construction is inadequacy so a new building system was developed at the beginning of the 20th century. So we have to use different alternative materials to decrease the concrete portion and to save the environment [1]. A Sandwich is a new

construction technology which was used by William Fairbairn in the year 1849.

The sandwich panel system is used around the world for low rise and high rise commercial structures. According to the architecture and structural point of view, the precast concrete sandwich panels are widely used [2]. Pokharel and Mahendran [3] studied the buckling effects of Australian sandwich plates by experimental and FE analysis with ABAQUS. Pokharel and Mahendran [4] studied local buckling effects of sandwich panel with Polystyrene foam core and steel faces subjected to bending and axial compression. Taczala and Banasiak [5] investigated the buckling of I-core panels which are subject to compressive loading with local buckling modes. Elasticity solution for buckling of sandwich panels under the uniaxial buckling load was studied by Kardomateas [6]. He derived the pre-buckling solution for the orthotropic phases for face and core. Kolsters and Zenkert [7] studied buckling of laser-welded sandwich panels using a finite element (FE) software ABAQUS. Indentation of aluminium foam core sandwich panel was investigated using finite element model developed in ABAQUS by Xu et al [8]. They compared their simulation results and experimental results with various thickness of foam and bottom face constrained. Chakrabarti et al. [9] presented buckling analysis of soft-core sandwich beam using the co-efficient FE model and calculated critical buckling loads for multi-layered sandwich beam having soft-core. The behavior of sandwich structures [10-11] and recent FE studies [12-25] have been studied by many researchers.

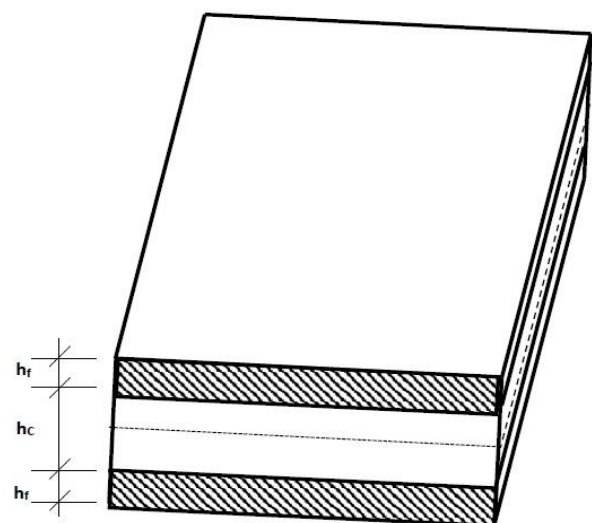


Fig. 1: Sandwich panel

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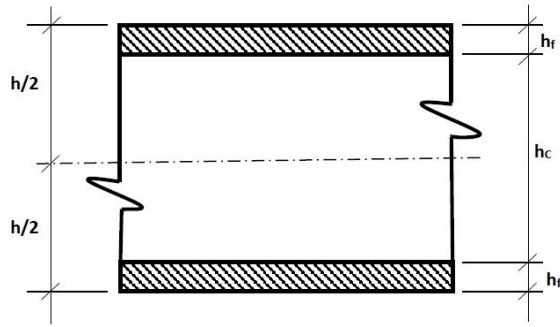


Fig. 2 Cross-section of sandwich panel

$$\{K\} = \begin{Bmatrix} K_x \\ K_y \\ K_z \\ K_{xy} \\ K_{yz} \\ K_{zx} \end{Bmatrix} = \begin{Bmatrix} \frac{\delta\theta_x}{\delta x} \\ \frac{\delta\theta_y}{\delta y} \\ 0 \\ \frac{\delta\theta_x}{\delta y} + \frac{\delta\theta_y}{\delta x} \\ \frac{\delta\theta_x}{\delta y} \\ \frac{\delta\theta_y}{\delta x} \end{Bmatrix}$$

Research Significance

Sandwich panels having isotropic core and orthotropic faces are lightweight, stiff and have good thermal properties. The sandwich panel absorbs heat and it also acts as an insulation material. Sandwich panel is having similar properties as compared to precast wall panels in terms of cost, durability, fire resistance and it can also be used as a shear wall, retaining wall and bearings walls. It can be used in the building expansion if needed. It is light in weight and easy to construct and it reduces the usage of concrete. Thus, for the safe design of the sandwich structure, an FE model is required to solve various buckling problems with precision and with a less computational cost. And also the effect of end conditions, aspect ratio and thickness is beneficial for researchers and for designers.

II. METHODOLOGY

The mathematical model in ABAQUS is as follows:

$$\{A\} = \{B\} + z\{C\}$$

where $\{A\} = \begin{Bmatrix} u \\ v \\ w \end{Bmatrix}$; $\{B\} = \begin{Bmatrix} u_0 \\ v_0 \\ w_0 \end{Bmatrix}$ and $\{C\} = \begin{Bmatrix} \theta_x \\ \theta_y \\ \theta_z \end{Bmatrix}$

$\{A\}$ is displacement, $\{B\}$ is mid-plane displacement along with the (x, y, z) coordinates and $\{C\}$ is rotation.

The strain vector:

$$\{\varepsilon\} = \{\varepsilon_0\} + z\{K\}$$

$$\text{where } \{\varepsilon\} = \begin{Bmatrix} \varepsilon_x \\ \varepsilon_y \\ \varepsilon_z \\ \gamma_{xy} \\ \gamma_{yz} \\ \gamma_{zx} \end{Bmatrix}; \{\varepsilon_0\} = \begin{Bmatrix} \varepsilon_{x0} \\ \varepsilon_{y0} \\ \varepsilon_{z0} \\ \gamma_{xy0} \\ \gamma_{yz0} \\ \gamma_{zx0} \end{Bmatrix} = \begin{Bmatrix} \frac{\delta u_0}{\delta x} \\ \frac{\delta v_0}{\delta y} \\ 0 \\ \frac{\delta u_0}{\delta y} + \frac{\delta v_0}{\delta x} \\ \frac{\delta w_0}{\delta y} + \theta_y \\ \frac{\delta w_0}{\delta x} + \theta_x \end{Bmatrix} \text{ and}$$

The governing equation for buckling analysis:

$$\{[K] - \lambda[K]_G\} \{\delta\} = \{0\}$$

where $\{\delta\}$ is the nodal displacement vector, $[K]$ is the linear stiffness matrices, λ is the critical buckling load and $[K]_G$ is the geometric stiffness matrices.

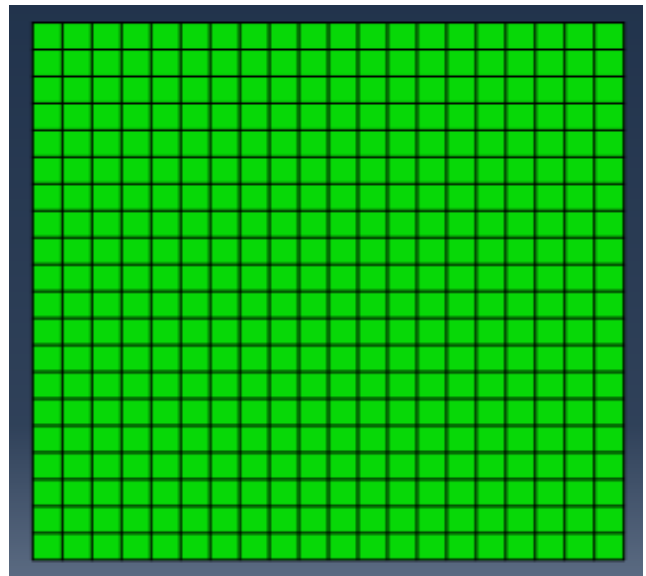


Fig. 3. The meshing of sandwich panel

III. RESULTS AND DISCUSSION

Convergence and Comparison Study:

The accuracy of the present FE model developed using ABAQUS has been validated with Kheirikhah et al. [26] [Table 1]. For convergence and comparison study a simply supported sandwich panel has been considered. The sandwich panel has been divided into 20 x 20 elements [Figure 3]. The material property and nondimensional formula have been taken from Kheirikhah et al. [26].



Table 1: Convergence and validation study

Mesh size	Non-dimensional Buckling load
12x12	5.9405
16x16	5.8021
18x18	5.7416
20x20	5.7280
Kheirikhah et al. [26]	5.6215

Present Study:

After validation effect of end condition, aspect ratio and thickness have been studied using the present FE model.

The material properties used for the analysis is

For face:

$$E_1 = 40E, E_2 = E, \mu_{12} = 0.25, G_{12} = 0.6, G_{23} = 0.5, G_{13} = 0.6$$

For core:

$$E_1 = E_2 = 0.04E, \mu = 0.250.$$

The buckling load parameter (non dimensional) used in the

$$\text{paper is } \frac{a^2 \lambda}{E_2 h^3}$$

where ‘a’ is length and ‘h’ is thickness of the sandwich panel.

Effect of end condition:

In this subsection, the effect of end conditions on the non-dimensional buckling load has been studied on two types of sandwich panels (0/90/C/90/0 and 0/C/0). The effect of six practically possible boundary conditions has been presented in figure 4. It can be seen that clamped sandwich panel have highest non-dimensional buckling compared to others for both types of sandwich panel. CFCF sandwich panel has the lowest non-dimensional buckling load.

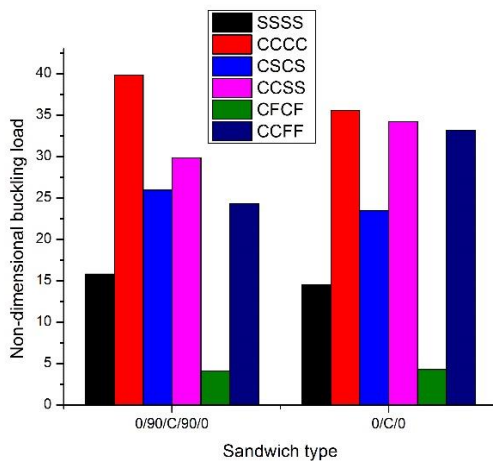


Fig. 4. Effect of end conditions on buckling of sandwich panels (a/b = 1 and a/h = 20).

[S – Simply supported; C – Clamped; F – Free]

Effect of aspect ratio:

In this subsection, the effect of a/b ratio on a non-dimensional buckling load has been studied on two types of sandwich panels (0/90/C/90/0 and 0/C/0). Here, a/b values have been varied from 1 to 5. It can be seen from figure 5 that with an increase in aspect ratio (a/b values) the non-dimensional buckling load increase for both types of sandwich panel.

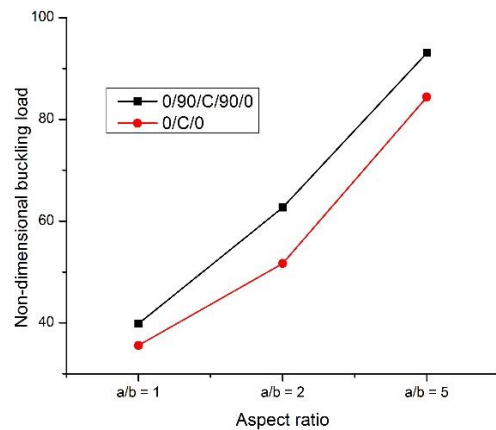


Fig. 5. Effect of aspect ratio on buckling of clamped sandwich panels (a/h = 20).

Effect of thickness:

In this subsection, the effect of a/h ratio on the non-dimensional buckling load has been studied on two types of sandwich panels (0/90/C/90/0 and 0/C/0). Here a/h has been varied from 20 to 100. It can be seen from figure 6 that with an increase in a/h ratio, non-dimensional buckling load of sandwich panel with isotropic core and orthotropic faces increases

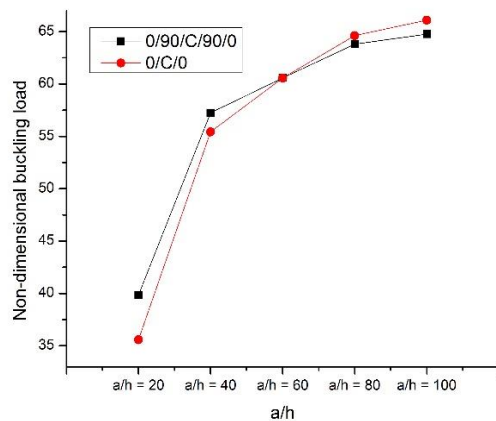


Fig. 6. Effect of a/h ratio on buckling of clamped sandwich panels (a/b = 1).

IV. CONCLUSIONS

An FE model has been developed in ABAQUS. Its validation has been done by comparing the results from the literature. The validation is clearly showing that the developed FE model is capable of solving various buckling problems with precision and with a less cost. In this work, buckling behavior of sandwich panel having isotropic core and orthotropic faces has been studied with different end conditions, aspect ratios and thickness ratios. The following general conclusions are:

- Clamped sandwich panels having isotropic core and orthotropic faces have the highest buckling load.



- With an increase in an aspect ratio of sandwich panel with isotropic core and orthotropic faces buckling load increases.
- With an increase in a/h ratio, buckling load of sandwich panel with isotropic core and orthotropic faces increases

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