

Noise Reduction of Deep Groove Ball Bearing (6205) by Process Optimization - An Experimental Approach

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Abstract: This paper presents a comprehensive study of the noise effect on deep groove ball bearing. Life of any rotatory components and machinery are depends on Ball Bearing so the bearing (DGBB) is crucial part of any rotary machinery and its failure causes disastrous failure of machinery. Noise level is the most important quality criteria of a ball bearing; it mainly depends on the following factors like Precision of the geometric forms (Track Profile, Curvature, and Talyrond etc.), Surface finish of the raceway and the balls, Cleanliness of the bearing, Type of lubricants etc. This paper mainly defines the experiment done on 18 random bearings, effect of different parameters of bearing to maintain a noise level by using DMAIC technology, and pareto chart after identification, reduction of noise has been done, which improves quality of bearing, cost effective & directly improves the quality of machine.

Index Terms: Curvature, Deep Groove Ball Bearing, DMAIC, Track Profile, Talyrond, Pareto Chart.

I. INTRODUCTION

Deep groove ball bearings are particularly versatile. They are suitable for high and very high speeds, accommodate radial and axial loads in both directions and require little maintenance. Deep groove ball bearings are the most widely used bearing type, they are available in many designs, variants and sizes; improvement in quality of DGBB increases rapidly, by optimizing the different cutting parameter, it improves the surface finish of deep groove ball bearing. The focused problem in this study is noise, and to eliminate this problem improvement in the surface texture of inner and outer track of the bearing is essential. Poor surface texture has become a big issue, especially for the automobile industry. DOE tool and DMAIC methodology is being used to define the method and technologies to identify the optimum parameter and its significant effects on ball bearing & on its component, after getting hypothetical result by using regression analysis the equation has been formed to identify the major optimum solution and reduce the level of noise [3].

II. METHODOLOGY

Noise level is the most important quality criteria of a ball

Bearing. It mainly depends on the following factors:

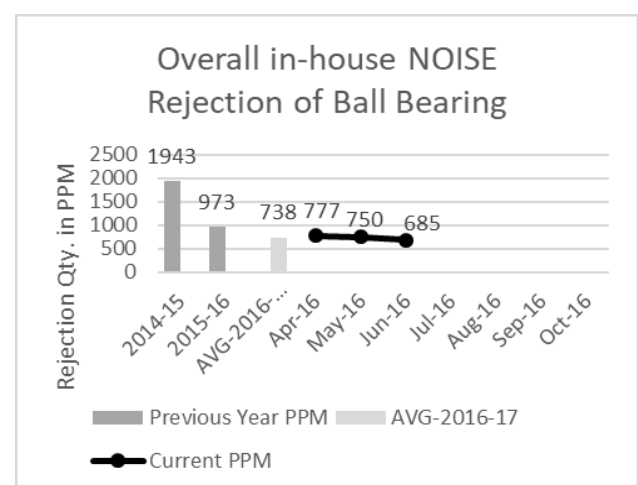
- Precision of the geometric forms (Track Profile, Curvature, Talyrond etc.).
- Surface finish of the raceway and the balls.
- Cleanliness of the bearing.
- Type of lubricant.

DMAIC methodology is a very common and successful 6-sigma technology to define the method and process to reduce the noise reduction in deep groove ball bearing. This section discuss reduction of noise rate in deep ball bearing by comparing Inner Race and Outer race of balls with different parameters (Ra, Rmax, Pt, Talyrond) at National Engineering Industry (NEI), Jaipur (India) using DMAIC cycle.

A. DEFINE PHASE

The main aim of this phase is to clearly solve the commercial problems; achieve goals, potential resources, venture possibility and high-level project timeline. As per venture contract, the core objective of this study is to reduce noise rate in ball bearing by improving quality and eliminating defects from 8373 PPM against target of 7716 PPM.

Graph A.a., shows Overall, in-house NOISE rejection of Assembled Ball Bearing in 2017-18, showing the reduction rate from July, 2015 to January, 2017 in parts per minute (PPM).



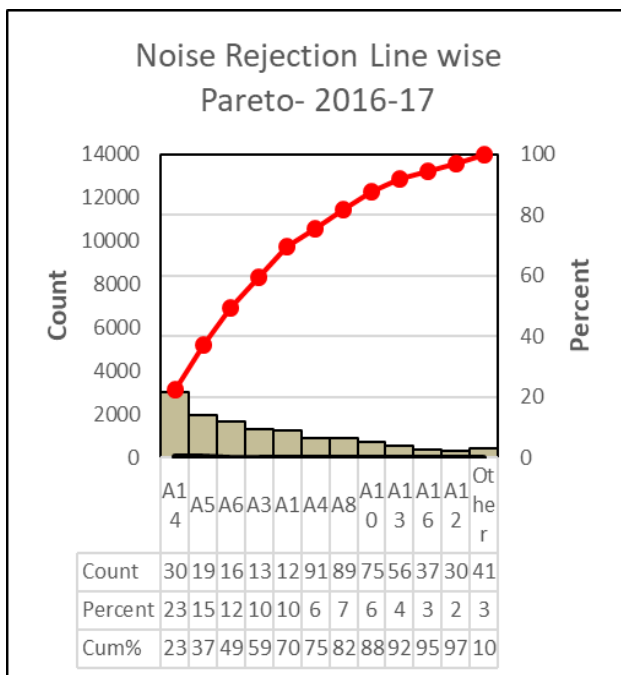
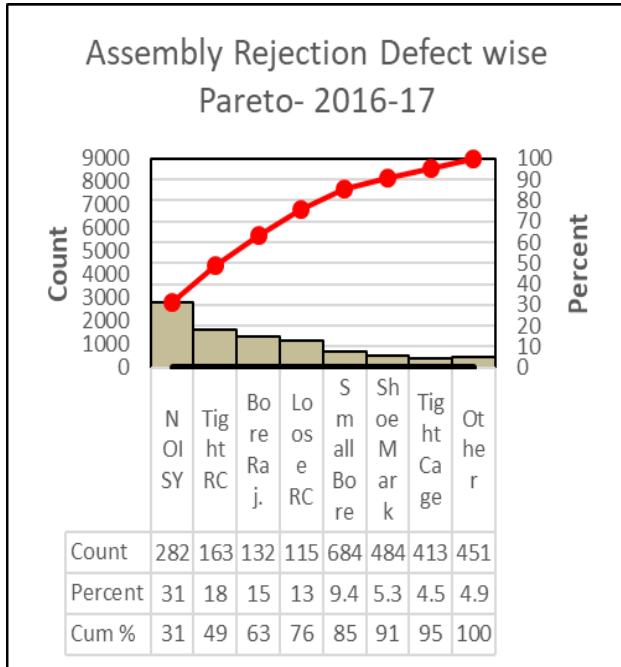
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Chart A. a. & A. b., Shows line wise Characteristics Pareto & Major characteristics.

Pareto graph is a column graph. Differentiating major factors contributing to problem from other factors, which have less contribution. Thus, it helps fixing priority to take first.



Inference: NOISE rejection is contributing 30.9% (2016-17) of overall assembly rejection.

Inference: A-14 line is contributing highest (23%) in overall assembly NOISE rejection.

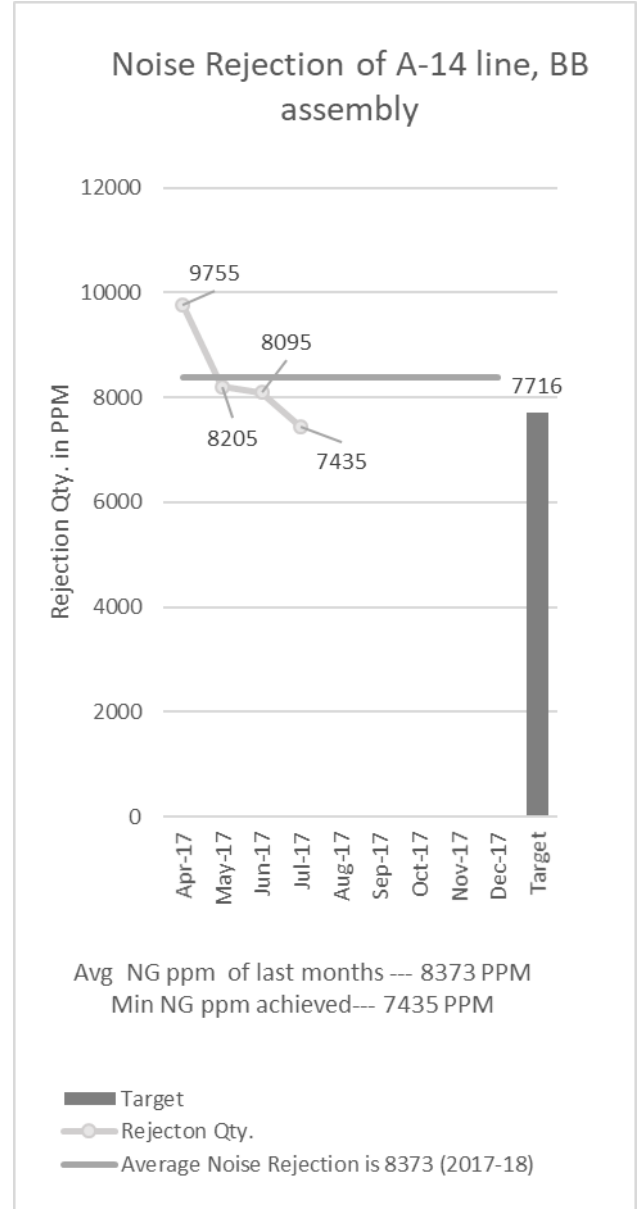
B. MEASURE PHASE

From the quality data of N.E.I. Ltd. for the pre-dispatch inspection (PDI), the first time through (FTT) trend chart and

the defect trend chart is generated.

The analyze phase segregates the top origins behind the defects produced with the help of fishbone (Ishikawa) diagram. [10]

Graph B.a., shows to reduce Assembled Bearing NOISE rejection from 8373 PPM to 7716 PPM on A-14 line.



Projected saving on A-14 line on target achievement

Average- 8373 PPM
Target- 7716 PPM
Saving- 657 PPM
Average monthly production = 1,30,000
Annual Production = 15,60,000

□ Annual rework saving
(657*15,60,000*9.25)/10,00,000 = Rs. 9481.00

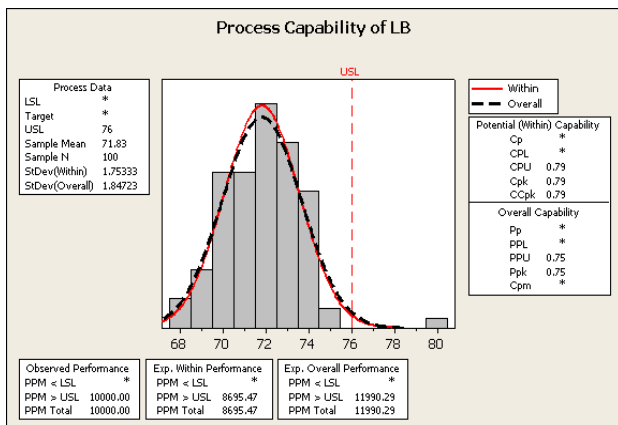
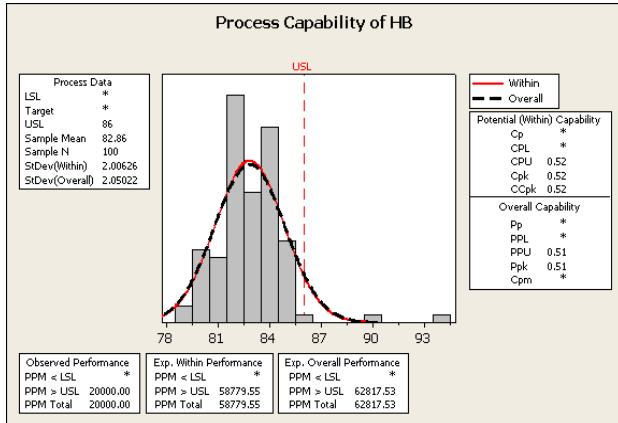


□ Profit Saving
(657*20*15, 60,000)/10, 00,000= Rs. 20,498.00

Total saving = Rs. 29,979 approx.

Projected saving on all line on target achievement
Annual saving = Rs. 6, 00,000 approx.

Chart B.a. & B.b., shows the Process Capability of High band and Low band.



Inference: Process capability is < 1.67 for HB value

Inference: Process capability is < 1.67 for LB value

Quick win opportunities

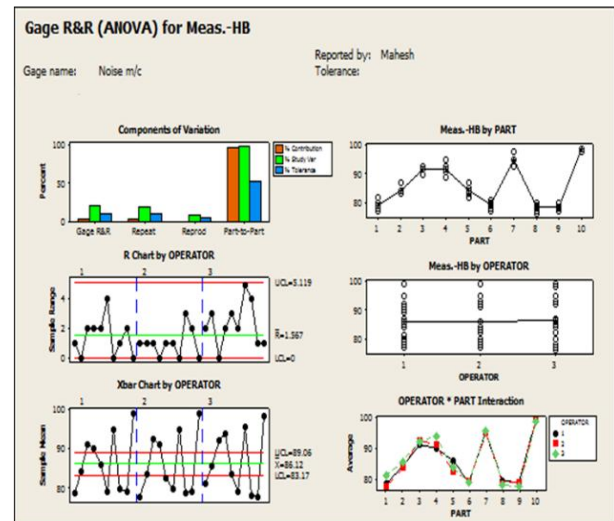
- Work head square ness of Outer race grinding m/c was high. It was find about 40µm against limit of max. 10µ Work head square ness corrected. Improvement in Track Talyrond observed. Yearly preventive check-sheet reviewed for taking care of this accuracy
- Riveting pressure was high. Cage marks observed in some balls in some bearings. Riveting pressure reduced from 4.6 to 3 kg/cm2.
- NOISE machine Spindle run out found unto 10µm.Spindle corrected; now spindle run out is less than 2 µm [4].

Gage repeatability and reproducibility (R&R), is a statistical tool used for the measurement of amount of variation system arising from the measurement device and the people taking the measurement.

Showing R&R graphs & readings for High Band & Low

Band by using ANOVA Tool.

Chart B.c. & B.d., shows the Gauge R&R High band.



Gage R&R

Source	VarComp	%Contribution (of VarComp)
Total Gage R&R	2.4889	4.11
Repeatability	2.0833	3.44
Reproducibility	0.4056	0.67
OPERATOR	0.0000	0.00
OPERATOR*PART	0.4056	0.67
Part-To-Part	58.0981	95.89
Total Variation	60.5870	100.00

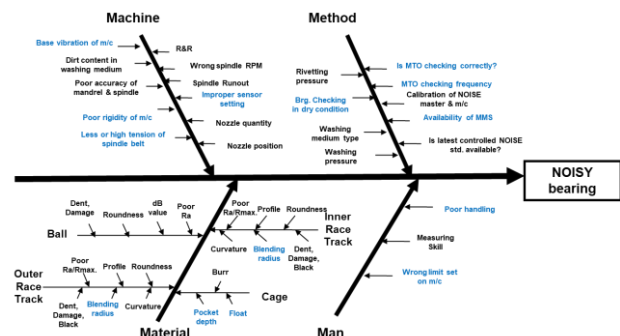
Source	StdDev (SD)	Study Var (6 * SD)	%Study Var (%SV)	%Tolerance (SV/Toler)
Total Gage R&R	1.57762	9.4657	20.27	11.01
Repeatability	1.44338	8.6603	18.54	10.07
Reproducibility	0.63683	3.8210	8.18	4.44
OPERATOR	0.00000	0.0000	0.00	0.00
OPERATOR*PART	0.63683	3.8210	8.18	4.44
Part-To-Part	7.62221	45.7333	97.92	53.18
Total Variation	7.78377	46.7026	100.00	54.31

Number of Distinct Categories = 6

Inference: Gage R&R for High Band is OK; this is not the cause of problem.

A fishbone diagram, also called a cause and effect diagram or Ishikawa diagram, is a visualization tool for categorizing the potential causes of a problem in order to identify its root causes, 14 factors are mainly responsible out of 42.

Diagram B.a., shows the Cause-Effect Diagram (Fish Bone) NOISE [3].



Inference: Number of responsible factor found are 14.

Table B.a., Shows the Cause-Effect Matrix (NOISE)



Cause-Effect Matrix - NOISE			NOISY Bearing	
	Customer Priority	10		
Process Step	Process Input			Total
1	Outer Races	O/R Track Surface finish	9	90
2	Outer Races	O/R Track Rmax	9	90
3	Outer Races	O/R Track Talyrond	9	90
4	Inner Races	I/R Track Surface finish	9	90
5	Inner Races	I/R Track Rmax	9	90
6	Inner Races	I/R Track Talyrond	9	90
7	Outer Races	O/R Track Profile	9	90
8	Outer Races	O/R Track Curvature	9	90
9	Outer Races	O/R Visual	9	90
10	Inner Races	I/R Track Profile	9	90
11	Inner Races	I/R Track Curvature	9	90
12	Inner Races	I/R Visual	9	90
13	Balls	Ball Talyrond	9	90
14	Balls	Ball Surface roughness	9	90
15	Balls	Balls Visual	9	90
16	Washing	Dirt content	9	90
17	Balls	Ball dB value	9	90
18	Cage	Appearance	9	90

Ranking scale 1 – 10, here only those factors has been shown which are on priority after filtering

Inference: Total no. of inputs found is 18 for NOISY bearings.

C. ANALYZE PHASE

Process / Product Failure Modes and Effects Analysis (FMEA) is define in analyze phase, the goal of the DMAIC analyze is to recognize the possible root causes for the process problem being addressed and then confirm actual root causes. This is very essential to rectify the problem to eliminate it. [4, 13]

Table C.a., Process / Product Failure Modes and Effects Analysis (FMEA)-Ball Bearing Assembly.

Process Function	Potential Failure Mode	Potential Effects of Failure	SEV	Potential Cause(s)/ Mechanism(s) of Failure	OCC	Current Process Controls	Current Process Control detection	DET	RPN
Outer Track Ra	Roughness out of specification	Abnormal Vibration & NOISE	6	improper Track blending chamfers, machining allowances	10	--	- Lot audited as per sample plan - 100% Noise checking of Bearing in Assembly	6	360

Outer Track Rmax	Outer Track Profile	Outer Race
Roughness out of specification	Profile value out of specification	Profile value out of specification
Abnormal Vibration & NOISE	Abnormal Vibration & NOISE	Abnormal Vibration & NOISE
6	6	6
improper Track blending chamfers, machining allowances	Improper Track blending chamfers, machining allowances	Incomplete superfine honing : stone broken/ Honing stone worn out
10	10	10
--	--	--
- Lot audited as per sample plan - 100% Noise checking of Bearing in Assembly	- Lot audited as per sample plan - 100% Noise checking of Bearing in Assembly	Stone wear signals are provided on Superfinishing machines - Lot audited as per sample plan - 100% Noise checking of Bearing on Noise Tester in Assembly
6	6	6
360	360	360

Inference: There are 12 numbers of causes whose RPN is >75

Table C.b. shows the Factor wise tool to be used



S. no.	Factor (X)	Data Type	Output (Y)	Data Type	Tool to be used
1	Outer Race Ra	C	NOISE	C	Fitted line plot
2	Outer Race Rmax.	C	NOISE	C	Fitted line plot
3	Outer Race Pt	C	NOISE	C	Fitted line plot
4	Outer Race Talyrond	C	NOISE	C	Fitted line plot
5	Outer Race Visual	D	NOISE	C	Interval Plot
6	Inner Race Ra	C	NOISE	C	Fitted line plot
7	Inner Race Rmax.	C	NOISE	C	Fitted line plot
8	Inner Race Pt	C	NOISE	C	Fitted line plot
9	Inner Race Talyrond	C	NOISE	C	Fitted line plot
10	Inner Race Visual	D	NOISE	C	Interval Plot
11	Ball Visual	D	NOISE	C	Interval Plot
12	Dirt Content	D	NOISE	C	Interval Plot

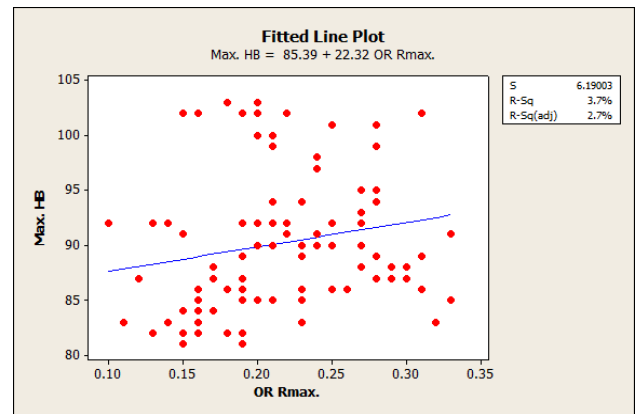
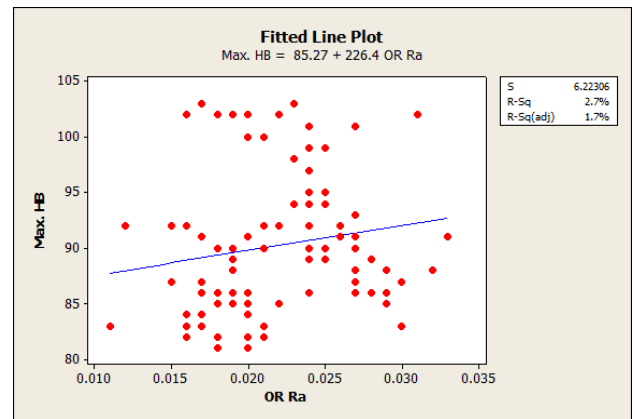
Data collection plan was made for 12 factors & appropriate graphical tools selected.

Table C.c. Shows Data Collection Check Sheet

Qty.	Brg. No.	NOISE				Inner Track	Outer Track	Ball	Dirt Content	Cage
		One side		Other side						
		HB	LB	HB	LB					
		Spec.	86	76	86					
1	1	88	88	88	88	OK	OK	OK	OK	OK
2	2	85	85	85	85	OK	OK	OK	OK	OK
3	3	94	94	94	94	OK	OK	OK	OK	OK
4	4	99	99	99	99	OK	OK	OK	OK	OK
5	5	94	94	94	94	OK	OK	OK	OK	OK
6	6	99	99	99	99	OK	OK	OK	OK	OK
7	7	103	103	103	103	OK	OK	OK	OK	OK
8	8	98	98	98	98	OK	OK	OK	OK	OK
9	9	103	103	103	103	OK	OK	OK	OK	OK
10	10	89	89	89	89	OK	OK	OK	OK	OK
11	11	103	103	103	103	OK	OK	OK	OK	OK
12	12	93	93	93	93	OK	OK	OK	OK	OK
13	13	86	86	86	86	OK	OK	OK	OK	OK
14	14	102	102	102	102	OK	OK	OK	OK	OK
15	15	98	98	98	98	OK	OK	OK	OK	OK
16	16	80	80	80	80	OK	OK	OK	OK	OK
17	17	82	82	82	82	OK	OK	OK	OK	OK
18	18	88	88	88	88	OK	OK	OK	OK	OK
19	19	88	88	88	88	OK	OK	OK	OK	OK
20	20	87	87	87	87	OK	OK	OK	OK	OK

Terms: R_a - Surface Roughness, P_t - Profile, R_{max} . Max. Roughness, NG- Grinding.

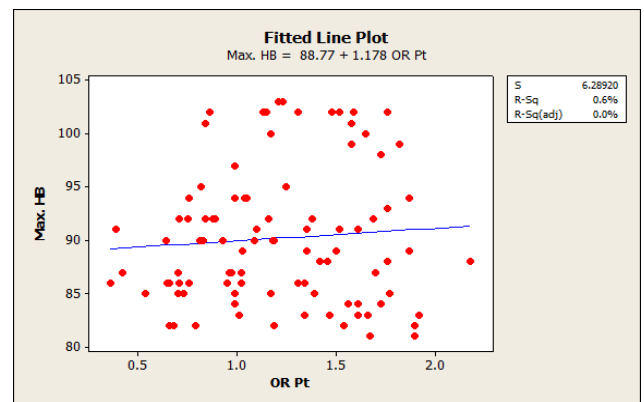
Graph C.a. & C.b. shows Analysis of Outer Race Ra v/s Noise Grinding & Outer race Rmax. V/s Noise Grinding



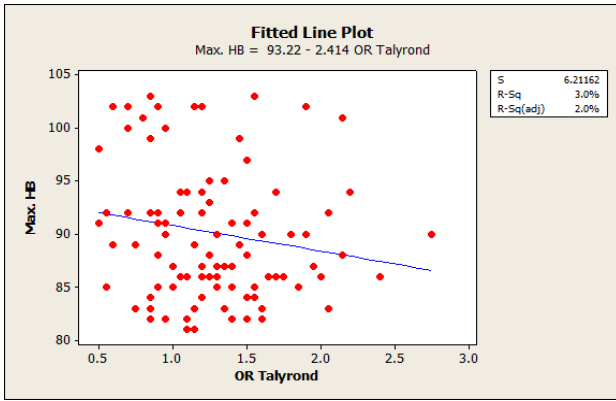
Inference: There is no relation between Outer Ra and NOISE NG.

Inference: There is no relation between Outer Rmax and NOISE NG.

Graph C.c & C.d. shows Analysis of Outer Race Profile v/s Noise Grinding & Outer race Talyrond v/s Noise Grinding



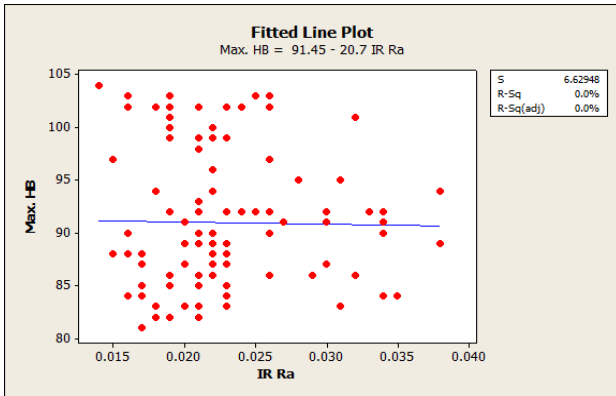
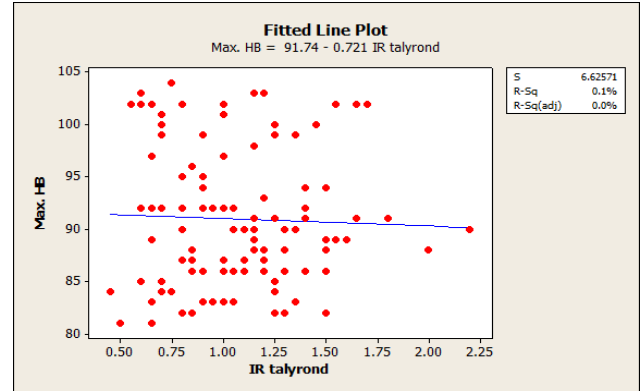
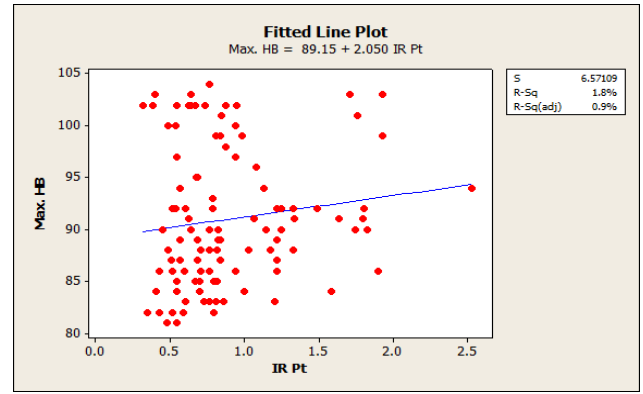
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Inference: There is no relation between Outer Race Profile v/s Noise Grinding.

Inference: There is no relation between Outer races Talyron v/s Noise Grinding.

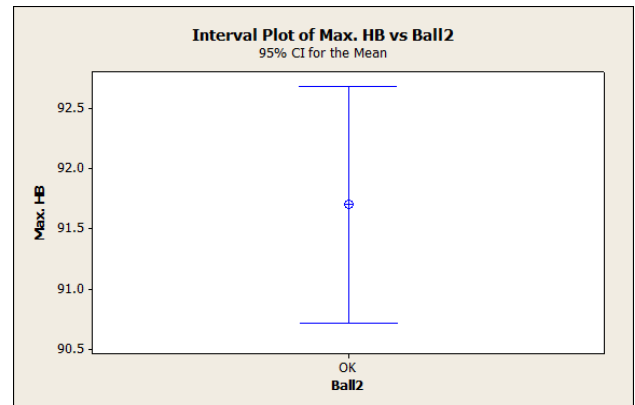
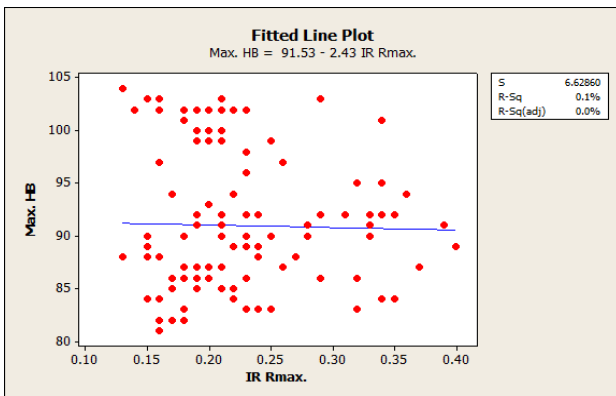
Graph C.e. & C.f. shows Analysis of Inner Race Surface Roughness v/s Noise Grinding & Inner race Maximum surface roughness V/s Noise Grinding.



Inference: There is no relation between Inner Race Profile v/s Noise Grinding.

Inference: There is no relation between Inner Race Talyron v/s Noise Grinding.

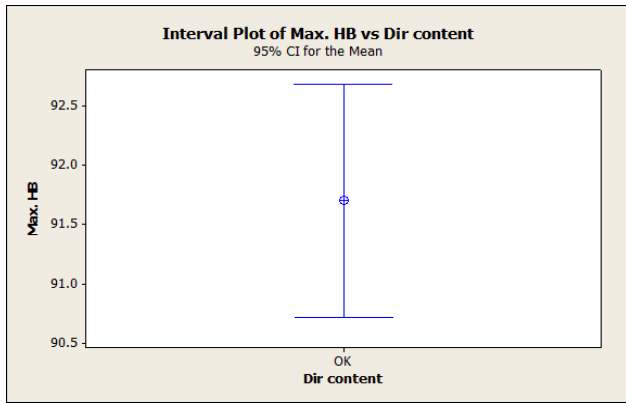
Graph C.i. & C.j. shows Analysis of Ball v/s Noise NG & Dirt content v/s Noise Grinding.



Inference: There is no relation between Inner Surface Roughness value and NOISE Grinding.

Inference: There is no relation between Inner Maximum Roughness value and Noise Grinding.

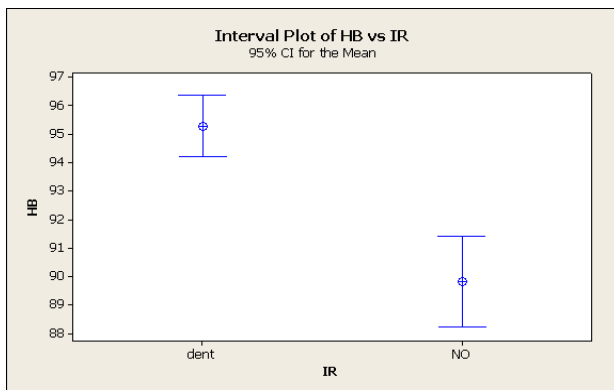
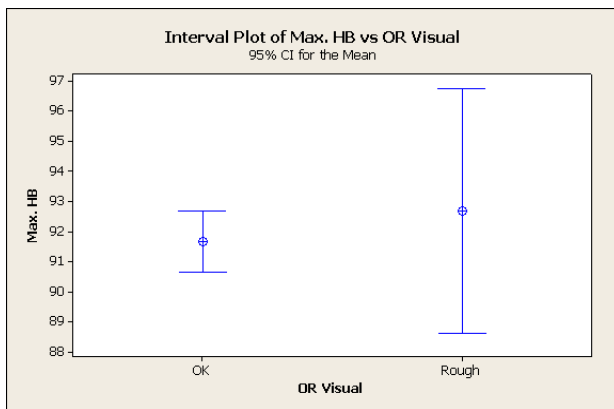
Graph C.g. & C.h. shows Analysis of Inner race Profile v/s Noise Grinding & Inner race Talyron v/s Noise Grinding.



Inference: There is no relation between Ball v/s Noise Grinding.

Inference: There is no relation between Dirt content v/s Noise Grinding.

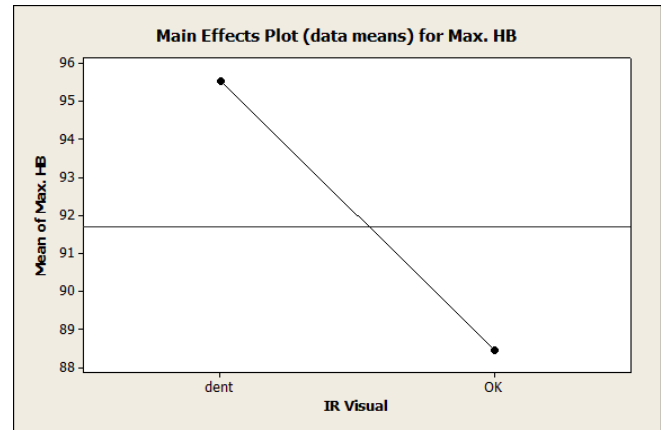
Graph C.k. & C.l. Shows Analysis of Outer Race visual v/s Noise Grinding & Inner Race Visual v/s Noise Grinding.



Inference: There is no relation between OR visual v/s Noise NG.

Inference: There is no relation between Inner race visual v/s Noise Grinding.

Graph C.m. Shows Analysis of Inner Race Track Dent v/s Noise Grinding Main effect plot.



Inference: There is a relation between Inner race dent and NOISE Grinding

Inference: Dent on Inner Race track will lead to NOISE rejection

Two-Sample T-Test and CI: NOISE, IR visual
Two-sample T for NOISE, shown in Table below.

IR visual	N	Mean	StDev	SE Mean
dent	25	97.56	3.01	0.60
no dent	25	84.48	2.04	0.41

Difference = μ (dent) - μ (no dent)

Estimate for difference: 13.0800

95% CI for difference: (11.6099, 14.5501)

T-Test of difference = 0 (vs not =): T-Value = 17.96.

P-Value = 0.000 DF = 42

P-Value = 0.000 DF = 42 this factor is significant.
Therefore, for NOISE rejection significant factor is Inner Track Visual (dent on Track).

D. IMPROVE PHASE

The main goal of the improve phase is to recognize the solution to the problem that the project aims to address. Identify inventive solutions to eradicate the key root causes in order to fix and avert method problems. 1-way ANOVA for Inner Race Visual v/s HB value is the exact technique to solve this problem. Therefore, counteractive actions are taking for root causes [10].

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Table D.a. Shows 1-way ANOVA for Inner Race Visual v/s High Band value or Confirmative Experiment for Inner Race Visual Grinding and OK for High Band value.

One-way ANOVA: HB. Versus Dent.					
Source	DF	SS	MS	F	P
Dent.	1	512.0	512.0	48.00	0.000
Error	6	64.0	10.7		
Total	7	576.0			
S = 3.266 R-Sq = 88.89% R-Sq(adj) = 87.04%					
Individual 95% CIs For Mean Based on Pooled StDev					
Level	N	Mean	StDev	-----+-----+-----+-----+--	
NO	4	83.50	1.29	(-----*-----)	
YES	4	99.50	4.43	(-----*-----)	
-----+-----+-----+-----+--					
		84.0	90.0	96.0	102.0
Pooled StDev = 3.27					

Inference: If there is Dent on Inner racetrack, then it will lead to NOISE rejection with HB value in range of 96 ~ 102 HB. Therefore, there should not be any dent on Inner Race Track surface, otherwise bearing will be rejected in NOISE.

E. CONTROL PHASE

Control phase is about to maintain the changes made in the improve phase. To ensure continued and sustainable success it will work. For controlling Update data's, documents, business process and training records are properly required. A Control chart can be useful during the control stage to evaluate the constancy of the enhancements over time by allocation as a guide to continue monitoring the process and provide a response plan for each of the measures being supervise in case the process becomes unstable [8,10].

Control action plan:

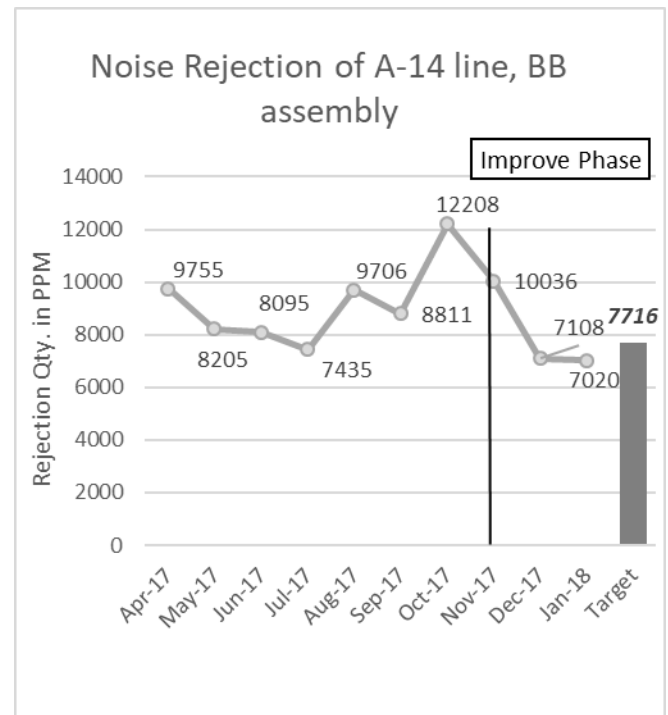
- Slide plate of Inner Race bore grinding m/c changed.
- Tool verification system for the grinding line developed.
- Inlet Channel of Inner Race ATDC gauge modified (Width & Slope reduced, and made straight).
- Stoppers in conveyors after IR Track grinding & PP gauge replaced from Metallic to soft material Hylem.
- Washing m/c after super finishing (Both IR & OR) replaced by improved washing m/c for effective washing.

III. RESULT

A Phase Summary is defined in the form of DMAIC (Define, Measure, Analyze, Improve and Control) methodology to show the process of reducing noise rejection of DGBB by optimizing cutting parameters. However, after experiments & graphical representation in Pareto chart and

regression analysis (MINITAB) the result shows that the dents occurred in inner racetrack are responsible for rapidly increasing noise. After taking corrective actions, the rejections were within acceptable limit.

Graph 3.a. shows Noise Rejection Graph



IV. CONCLUSION

Now a day, Six Sigma is a very common theme followed in industries to increase the productivity and reduce defect rates. In this study DMAIC cycle has been utilized to reduce the noise reduction rate in ball bearing industry. As per the first stage of DMAIC cycle, it has been defined that the noise generated by ball bearing should be minimized. In the second stage, all the data of pre-dispatch inspection (PDI) for first time through has been collected and Pareto analysis was applied to get the major faults with the help of Ishikawa diagrams the root causes of major defects. In fourth stage, the main root causes were eradicated to prevent the process problems. In last stage, the changes made in improve stage have been continued exploiting control values.

- Noise Rejection in 2015-167151 ppm
- From Sep-16 to Jan-17 1164 ppm
- Rejection Reduced 5987 ppm

Total production from Jul-16 to Jan-17
(Project started in Jul-16) 13,35,722

Bearing Saved in no. 7,997
Total cost @ Rs. 14/- per Brg. 1,11,958



Thus, after conducting the DMAIC cycle, major parameters like Surface finish, Surface roughness, profile and Talyrond were compare with High band and low band of ball bearing and then the reason found that due to inner dent the noise was created now tool verification system is made for all grinding lines to verify all tooling's at a particular frequency, roundness, surface roughness and high band were almost near the standard limits. Hence, Bearing NOISE rejection rate is improved and the noise rejection is now 7716 (2016-17) ppm from 8373 ppm, saving is 657ppm.

ACKNOWLEDGMENT

It is optional. The preferred spelling of the word "acknowledgment" in American English is without an "e" after the "g." Use the singular heading even if you have many acknowledgments. Avoid expressions such as "One of us (S.B.A.) would like to thank" Instead, write "F. A. Author thanks" *Sponsor and financial support acknowledgments are placed in the unnumbered footnote on the first page.*

REFERENCES

1. Sound and Vibration in Rolling Bearings, Tatsunobu Momono and Banda Noda Basic Technology Research and Development Center, NSK Motion & Control No. 6 — 1999.
2. Dynamic model of a deep-groove ball bearing including localized and distributed defects, Dynamic model of a deep-groove ball bearing including localized and distributed defects. Part 1: Theory, J Sopanen, A Mikkola, Proceedings of the Institution of Mechanical Engineers, Part K: Journal of Multi-body Dynamics, Vol 217, Issue 3, 2003.
3. To develop the model for improving the surface texture of deep groove ball bearing to optimize cutting parameter, by using the methodology of Design of Experiment, Shradha Arya1 , M.P. Singh2 , Manish Bhargava3, Journal of Emerging Technologies and Innovative Research (JETIR) volume 5, issue 2, 2018.
4. Experimental Study for Vibration Behaviors of Locally Defective Deep Groove Ball Bearings under Dynamic Radial Load, V. N. Patel,1 N. Tandon,2 and R. K. Pandey3, Advances in Acoustics and Vibration, 7 pages, Volume 2014.
5. Using Ceramic Balls to Reduce Noise in a Linear Guideway Type Recirculating Linear Ball Bearing, Hiroyuki Ohta, Takumi Nakagawa, [Journal of Tribology], Journal of Tribology | Volume 125 | Issue 3, J. Tribol 125(3), 480-486 (Jun 19, 2003) (7 pages).
6. Improving Quality Parameters of Steel Balls Affecting Noise in Ball Bearing Chirag Ashok kumar Gandhi1, Dr. K. N. Vijaykumar2, Frank Crasta3, International Research Journal of Engineering and Technology (IRJET) e-ISSN: 2395 -0056 Volume: 03 Issue: 06 | June-2016.
7. Prediction of Defects in Antifriction Bearings using Vibration Signal Analysis, M Amarnath, R Shrinidhi, A Ramachandra, S B Kandagal, IE(I) Journal-MC, Vol 85, July 2004.
8. Dynamic response of an unbalanced rotor supported on ball bearings, Journal of Sound and Vibration, M. Tiwari, K. Gupta, O. Prakash, 238 (2000) 757–779.
9. Dynamic model of a deep-groove ball bearing including localized and distributed defects. Part 2: Implementation and results , J Sopanen, A Mikkola, , Proceedings of the Institution of Mechanical Engineers, Part K: Journal of Multi-body Dynamics, September 1, 2003.
10. Quality Improvement Methodologies – PDCA Cycle, RADAR Matrix, DMAIC and DFSS, M. Sokovic, D. Pavletic, K. Kern Pipan, Journal of Achievements in Materials and Manufacturing Engineering 43/1 (2010) 476-483.
11. An Overview of Bearing Vibration Analysis, Dr. S. J. Lacey, Nov/Dec 2008 ME | maintenance & asset management vol 23 no 6.
12. Mathematical Model of Load Distribution in Rolling Bearing, Tatjana Lazovic, Mileta Ristivojevic, Radivoje Mitrovic, FME Transactions VOL. 36, No 4, 2008 • 191
13. A theoretical model to predict the vibration response of rolling bearings in a rotor bearing system to distributed defects under radial load, Tandon N , Choudhury A. , Journal of Tribology. 2000;122(3):p. 609-615.
14. Vibration signature analysis for Ball Bearing of Three Phase Induction Motor , Amit Shrivastava1, Dr. Sulochana Wadhvani2, IOSR Journal of Electrical and Electronics Engineering (IOSRJEEE) ISSN: 2278-1676 Volume 1, Issue 3 (July-Aug. 2012), PP 46-50.
15. Analysis of deep groove ball bearing using fema review , Sumit Kumar Dahiya, A.K. Jain, International Journal of Current Research and Review www.ijcrr.com Vol. 04 issue 16 Aug 2012.
16. Modified Prioritization Methodology for Risk Priority Number in Failure Mode and Effects Analysis , N. Sellappan, K. Palanikumar , International Journal of Applied Science and Technology Vol. 3 No. 4; April 2013.
17. Modeling of the Behavior of a Deep Groove Ball Bearing in Its Housing , Ayao E. Azianou, Karl Debray, Fabrice Bolaers, Philippe Chiozzi, Frédéric Palleschi , DOI: 10.4236/jamp.2013.14009 PP.45 - 50, Pub. Date: November 6 , 2013.
18. Vibration Analysis of Ball Bearing , Ravindra A.Tarle1, Nilesh K.Kharate2, Shyam P. Mogal3, International Journal of Science and Research (IJSR) ISSN (Online): 2319-7064 Index Copernicus Value (2013): 6.14 | Impact Factor 4.438.
19. Identification Of Dynamic Rigidity For High Speed Spindles Supported On Ball Bearings , Jakeer Hussain Shaik1, Srinivas J 2, Ijret: International Journal Of Research In Engineering And Technology Eissn: 2319-1163 | Pissn: 2321-7308,2013.
20. Defect Detection in Deep Groove Polymer Ball Bearing Using Vibration Analysis, Attel Manjunath1 and D V Girish2, International Journal of Recent Advances in Mechanical Engineering (Ijmech) Vol.2, No.3, August 2013. G. O. Young, "Synthetic structure of industrial plastics (Book style with paper title and editor)," in Plastics, 2nd ed. vol. 3, J. Peters, Ed. New York: McGraw-Hill, 1964, pp. 15–64.
21. W.-K. Chen, Linear Networks and Systems (Book style). Belmont, CA: Wadsworth, 1993, pp. 123–135.
22. H. Poor, An Introduction to Signal Detection and Estimation. New York: Springer-Verlag, 1985, ch. 4.
23. B. Smith, "An approach to graphs of linear forms (Unpublished work style)," unpublished.
24. E. H. Miller, "A note on reflector arrays (Periodical style—Accepted for publication)," IEEE Trans. Antennas Propagat., to be published.
25. J. Wang, "Fundamentals of erbium-doped fiber amplifiers arrays (Periodical style—Submitted for publication)," IEEE J. Quantum Electron., submitted for publication.
26. C. J. Kaufman, Rocky Mountain Research Lab., Boulder, CO, private communication, May 1995.
27. Y. Yorozu, M. Hirano, K. Oka, and Y. Tagawa, "Electron spectroscopy studies on magneto-optical media and plastic substrate interfaces(Translation Journals style)," IEEE Transl. J. Magn.Jpn., vol. 2, Aug. 1987, pp. 740–741 [Dig. 9th Annu. Conf. Magnetics Japan, 1982, p. 301].
28. M. Young, The Technical Writers Handbook. Mill Valley, CA: University Science, 1989.
29. (Basic Book/Monograph Online Sources) J. K. Author. (year, month, day). Title (edition) [Type of medium]. Volume(issue). Available: [http://www.\(URL\)](http://www.(URL))
30. J. Jones. (1991, May 10). Networks (2nd ed.) [Online]. Available: <http://www.atm.com>
31. (Journal Online Sources style) K. Author. (year, month). Title. Journal [Type of medium]. Volume(issue), paging if given. Available: [http://www.\(URL\)](http://www.(URL))
32. R. J. Vidmar. (1992, August). On the use of atmospheric plasmas as electromagnetic reflectors. IEEE Trans. Plasma Sci. [Online]. 21(3). pp. 876–880. Available: <http://www.halcyon.com/pub/journals/21ps03-vidmar>

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