

Tuning of PID Controller for superheated Steam Temperature System using Modified Zeigler-Nichols Tuning Algorithm

A. Yasmine Begum, G.V. Venkata Marutheswar, K. Ayyappa Swamy

Abstract— This paper explains the tuning of superheated steam temperature system using Modified Zeigler tuning Algorithm. PID control has a wide range of applications in industrial control. Since many process plants use PID to control the dynamics involved in the process. It has been found possible to set satisfactory controller parameters from less plant information than a complete mathematical model. The fifth order model of super-heated temperature system is converted into first order model with delay using process reaction curve method. The analog PID controller is designed for superheated steam temperature system using Modified Zeigler tuning algorithm and Zeigler Nichols algorithm and results are compared.

Index Terms— Criterion of Optimality Tuning Rules, Figures of Merit, Process reaction curve.

I. INTRODUCTION

PID control has a wide spectrum of applications in industrial control. The choice of analog controllers is related to the model of the process to be controlled. Tuning is adjustment of parameters to achieve estimated response. The step response of the system should have minimum overshoot and one quarter-decay ratio and also include minimum rise time, and minimum settling time. The boiler has many variables to be precisely controlled for efficiency and safety. Among those variables, super-heated steam temperature is one of the important variables. The fifth order model of super-heated steam temperature is used for study [3], [4].

The analog controllers were tuned for delay plus first order transfer function model. The superheated steam temperature system is to be reduced from fifth order to first order transfer function model.

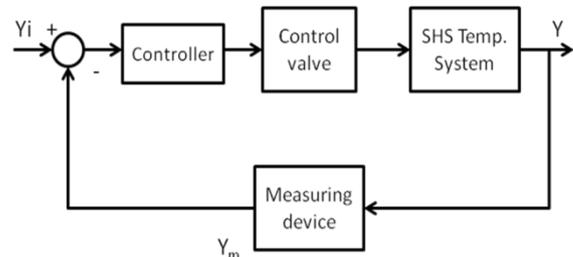


FIGURE 1: BLOCK DIAGRAM OF SHS TEMPERATURE CONTROL SYSTEM Formula to identify delay plus first order system [3], [4]:

$$G_n(s) = \frac{ke^{-Ls}}{Ts+1} \quad (1)$$

k - Gain

L - Time delay

Many industrial processes are nonlinear and thus it is complicate to describe mathematically. However, it is known that a good many nonlinear processes can be satisfactorily controlled using PID controllers provided controller parameters are tuned well.

The Transfer function of super heated temperature system [10]

$$G_p = 1.3882 / (19S+1)^5 \quad (2)$$

Therefore, the identified delay plus first order system using Process reaction curve method is

$$\frac{Ke^{-Ls}}{TS+1} = \frac{0.771167e^{-56.278S}}{42.934S+1}$$

A. PID Controller

PID controller calculates an error value as the difference between a measured processed variable and a desired set point the controller minimize the error by adjusting the process control inputs. The main structure of a PID control system is highlighted in Fig. 2, where it can be observed that in a PID controller, in order to generate the proportional, integral, and derivative actions, error signal e(t) is used as a result the signals weighted and summed to form the control signal u(t) which is applied to the plant model. [1],[2] The mathematical derivation of the PID controller is

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* Correspondence Author (s)

A.Yasmine Begum*, Department of EIE, Sree Vidyanikethan Engineering College, Tirupathi, India.

Dr.G.V.Marutheswar, Department OF EEE, Sri Venkateswara University, Tirupathi, India.

K.Ayyappa Swamy, Department of EIE, Sree Vidyanikethan Engineering College, Tirupathi, India.

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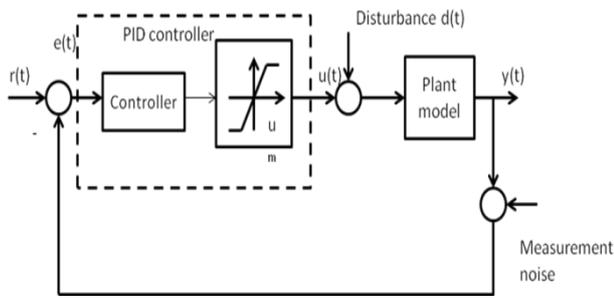


FIGURE 2: BLOCK DIAGRAM OF PID CONTROLLER

$$u(t) = K_p \left[e(t) + \frac{1}{T_i} \int_0^t e(\tau) d\tau + T_d \frac{de(t)}{dt} \right] \quad (4)$$

Where $u(t)$ is the input signal to the plant model, the error signal $e(t)$ is defined as $e(t) = r(t) - y(t)$, and $r(t)$ is the reference input signal.

B. PID Derivative Controller

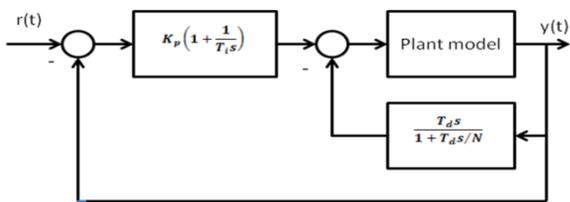
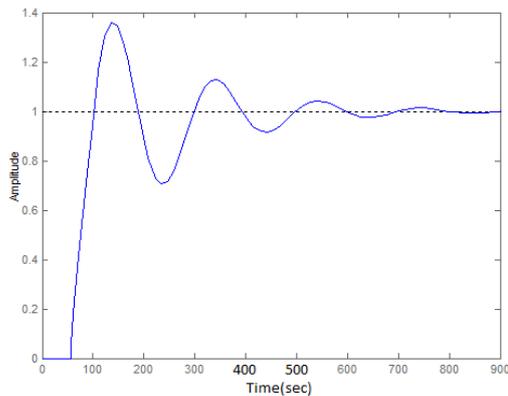


FIGURE 3: BLOCK DIAGRAM OF PID CONTROLLER WITH DERIVATIVE IN FEEDBACK

$$G_c(s) = K_p \left(1 + \frac{1}{T_i s} \right) \quad (5)$$



$$H(s) = \frac{(1+K_p/N)T_i T_d s^2 + K_p(T_i + T_d/N) + K_p}{K_p(T_i s + 1)(T_d s/N + 1)} \quad (6)$$

II. ZIEGLER-NICHOLS TUNING FORMULA

A. Empirical Ziegler-Nichols Tuning Formula

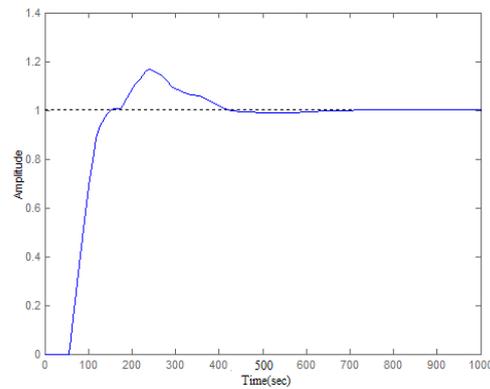
An important empirical tuning formula was proposed by Ziegler and Nichols in the earlier 1942. The tuning formula is acquired when the plant model [7,8] is presented by a first-order plus dead time (FOPDT) which can be expressed by

$$G_n(s) = \frac{ke^{-Ls}}{Ts+1} \quad (9)$$

In actual process control systems, many number of plants can be approximately modeled. If the system model can't be physically derived, experiments can be carried to bring out the parameters for the approximate model.[5],[6] For example, if the step response of the plant model is calculated through an experiment, the output signal will be recorded. from which the parameters of k , L , and T (or a , where $a = k/L$) can be brought out by process reaction curve method. Highly advanced curve fitting methods can also be used. With L and a , the Ziegler-Nichols formula in Table1 can be utilized to get the controller parameters.

TABLE1:FORMULA FOR PARAMETERS OF ANALOG CONTROLLERS BASED ON ZIEGLER-NICHOLS ALGORITHM

Controller type	from step response		
	K_p	T_i	T_d
P	$1/a$		
PI	$0.9/a$	$3L$	
PID	$1.2/a$	$2L$	$L/2$



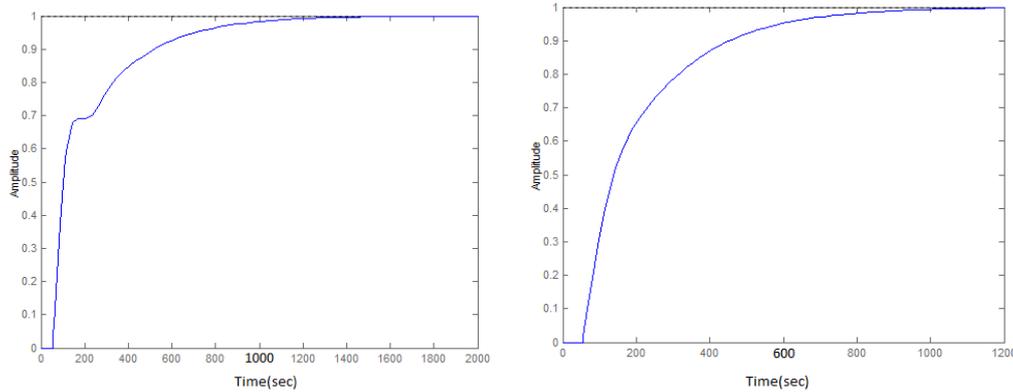


FIGURE 4: STEP RESPONSE OF PID CONTROLLER FOR (A) SHS SYSTEM USING ZEIGLER NICHOLS TUNING ALGORITHM, (B) DERIVATIVE IN FEEDBACK LOOP USING ZEIGLER NICHOLS TUNING ALGORITHM, (C) SHS SYSTEM USING MODIFIED ZEIGLER NICHOLS TUNING ALGORITHM ($\Phi_b=45^\circ$, $r_b=0.45$), (D) SHS SYSTEM USING MODIFIED ZEIGLER NICHOLS TUNING ALGORITHM ($\Phi_b=45^\circ$, $r_b=0.45$).

Modified Zeigler-Nichols Tuning Algorithm: The PID controller can be designed such that

$$K_p = \frac{r_b \cos(\phi_b - \phi_a)}{r_a} \quad (7)$$

$$\omega_o T_d - \frac{1}{\omega_o T_i} = \tan(\phi_b - \phi_a) \quad (8)$$

Clearly, T_i and T_d are not unique. To get a unique PID design, it is a usual practice to set $T_d = \alpha T_i$, where α is a constant. Given an α , T_i and T_d can be obtained uniquely from

$$T_d = \alpha T_i \quad (10)$$

By inspection, it is seen that the Ziegler–Nichols tuning formula is a special case when $\alpha = 1/4$. The Ziegler–Nichols tuning formula can be rewritten as follows:[8,9]

$$K_p = K_c r_b \cos \phi_b \quad (11)$$

$$T_i = \frac{T_c}{\pi} \left(\frac{1 + \sin \phi_b}{\cos \phi_b} \right) \quad (12)$$

$$T_d = \frac{T_c}{4\pi} \left(\frac{1 + \sin \phi_b}{\cos \phi_b} \right) \quad (13)$$

where $r_a = 1/K_c$, $\phi_a = 0$, and $\alpha = 1/4$.

It can be seen that the PI or PID controllers can be designed by a suitable choice of r_b and ϕ_b . The design problem is then one of selecting suitable values for these two parameters to give the appropriate performance. This is called a modified Ziegler–Nichols PI/PID tuning.[11,12]

Table2: Performance of PID Controller based on various tuning methods

Sl.No	Time Domain Specifications	PID controller Z-N settings	PID controller with derivative in feedback using Z-N settings	PID controller with modified Z-N settings	
				$\phi_b=20^\circ$, $r_b=0.2$	$\phi_b=45^\circ$, $r_b=0.45$
1	Rise Time	1.6924	2.8686	6.0835	3.9521
2	Settling Time	663.5979	397.5467	776.0153	954.6468
3	Settling Min	0.1813	0.1436	0.1901	0.3744
4	Settling Max	1.3579	1.1677	0.9997	1.0000
5	Overshoot	35.7860	16.7660	0	0
6	Undershoot	0	0	0	0
7	Peak	1.3579	1.1677	0.9997	1.0000
8	Peak Time	135.2266	239.8240	1.5933e+003	2.8747e+003

III. CONCLUSION

The delay plus first order system for super heated steam temperature control is identified using process reaction curve method .By comparing the responses of PID Controller tuning using Ziegler–Nichols tuning formula exhibits strong oscillation in the set-point response and a large overshoot. The response PID controller for super heated steam temperature system tuned using modified zeigler tuning algorithm has no overshoot and no oscillations. Hence it can

be concluded that the modified tuning method is advantageous over the original Ziegler–Nichols PI/PID tuning technique.

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A.yasmine Begum obtained B.E from M.A.M College of Engineering,Tirchy in 2005 and M.E from Satyabama University, Chennai in 2008 respectively.At present she is working as Assistant Professor in Sree vidyanikethan Engineering College,Tirupati.She is a Life member of ISOI.She is carrying out her Ph.d work in the area of Control systems.



Dr. G.V.Venkata Marutheswar obtained B Tech and M Tech from S V University in 1985 and 1988 respectively.He obtained Ph.d from S V University,Tirupati in 2009.At present he is working as Professor in S V University College of Engineering,Tirupati.He is a Life member of ISTE.His current interest include Fuzzy control systems and Neuro

Fuzzy control systems.



K.Ayyappa Swamy obtained B Tech from Aditya Engineering College in 2009 and M Tech from Gitam University in 2012. At present he is working as Assistant Professor in Sree Vidyanikethan Engineering College,Tirupati.