

Experimental Analysis of Hybrid Energy Operated Refrigerator Coupled In EV



Surender Kumar, Rabinder Singh Bharj

Abstract: This paper is focused on the performance of a solar-assisted DC refrigerator installed on the backside of the electric vehicle (EV). The experiments are performed by varying load conditions inside the refrigerator. The experimental setup consists of four solar PV panels, a charge controller, battery bank, voltage converter, DC refrigerator, and an electric vehicle. The temperature inside the refrigerator cabin was controlled with the thermostat position adjustment. The solar PV panels of the vehicle was generating 2.5-4 kWh energy on the average sunny day. The refrigerator's inside temperature was decreased with a faster rate at the third thermostat position and consuming higher energy at the seventh thermostat position among all load conditions. The fourth and fifth thermostat positions were better at maintaining the lower desired temperature inside the refrigerator cabin by consuming the minimum energy. The COP of the refrigerator was decreasing with the increasing compressor speed. The battery bank was able to run the refrigerator 240 hr, 96 hr, 72 hr for the no-load, 15 L load, and 25 L load conditions at the higher thermostat position. The vehicle was travelling 68.3 km, 65.3.6 km, 63.4 km distance in no-load, 100 kg, and 200 kg load conditions respectively by consuming 3010 Wh, 3230 Wh, and 3450 Wh energy. The travelling charge of this vehicle was 1-1.5 INR per kilometer.

Keywords: Hybrid Energy Refrigerator, DC Compressor, Temperature Variation, Photovoltaic (PV), EV performance.

I. INTRODUCTION

A. The Current Scenario in Perishable Foodstuff Transportation

More than 40% of food delivery items require refrigeration, but only 15% of products use this facility due to energy shortfall worldwide. The refrigerated transport system is the core of the cold chain. The supply of perishable food items at a distant location depends on the refrigerated food transportation system [1]. Refrigerated vehicles such as trucks and vans play an essential role in the localized delivery of perishable foodstuffs. Globally, about 4 million food transport vehicles are available in the current market. More than 65% of these vehicles are using for milk and its products transportation. Only 35% of vehicles are using for perishable

foodstuff transportation [2]. The shelf life and quality of perishable foodstuffs highly depend on the storage temperature range of 5 to -15 °C. Today perishable foodstuff transportation is facing sustainability problems, especially in the urban area of developing countries. About 20% more fuels are consuming by diesel engine vehicles integrated with the refrigeration system. These refrigerated vehicles are generating more GHG emissions [3-5]. Therefore, energy-efficient load-carrying vehicles are required in cold chain transportation (CCT) that produce the minimum amount of GHG emissions.

B. Energy Consumption and Food Loss in the CCT

Presently reliable and eco-friendly refrigeration system is needed in the CCT for temporary storage. Nearly 30% of foodstuffs are wasting in the food supply chain (FSC) before the customer receives it. The perishable foods are wasting on the farm level due to environmental change, less availability of refrigerated vehicles, poor management, lack of cold chain infrastructures, and economic problems [6-7]. More than 15% of total world energy is consumed in the refrigerated food supply chain (RFSC). This energy is generated by fossil fuels that again cause GHG emissions. The new fresh food demand is increasing worldwide in the FSC due to the increasing population and peoples' living standards in urban areas. Environmental conditions play a vital role in energy consumption and transportation cost in the FSC [8-12]. The quality decay of the food product starts if the perishable foodstuffs delivery time increases.

C. Refrigerated Electric Vehicle Role in Smart City Foodstuff Delivery

Currently, the transport sector releases 9000 billion tons of carbon dioxide (CO₂), which is expected to increase by 60% in 2050. The transportation sector oil demand continuously increases, which is a serious issue. The cost of fossil fuels grows continuously in the past decades. The solar-assisted refrigerating electric vehicle (SAREV) is operationally eco-friendly and energy-efficient, which is a good alternative compared to conventional vehicles [13]. The impressive growth in the food supply chain has been recorded due to urbanization. About 50% of the world population lives in metro cities. The frozen food market is growing, with a 7% CAGR globally [14-16]. By the use of refrigeration during perishable foodstuff transportation, the bacteria growth controls inside foodstuff are possible. The alternative SAREV is smaller in size and can be suitable for timely foodstuff delivery in congested city zones.

In the current scenario, many developing countries need a reliable and energy-efficient approach to inter-city refrigerated

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* Correspondence Author

Surender Kumar*, Research Scholar, Mechanical Engineering Department, National Institute of Technology, Jalandhar (Punjab), India.

Dr. R.S. Bharj, Associate Professor, Mechanical Engineering Department, National Institute of Technology, Jalandhar (Punjab), India.

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foodstuff transportation. This paper describes the experimentation on the refrigerator and electric vehicle that is used in the smart cold chain transportation.

Earlier reviews on the integral solar refrigerating system are briefed in Table 1.

Table- I: Review on Integral Solar Refrigerating System Performance

Year & reference	Type of compressor & working refrigerant	Method & experimental parameters
2009 [16]	AC compressor (110 W); R134a refrigerant.	<ul style="list-style-type: none"> • Solar operated domestic refrigerator (SDR); • The solar intensity with time, solar photovoltaic panel (SPV) current, and voltage; • Freezer temperature and ambient temperature; • COP of the refrigerator with time.
	<i>Short description:</i> The COP of the refrigerator was decreased with sunrise; the maximum COP was recorded 2.1 at 7 am.	
2013 [17]	12V DC compressor; R134a refrigerant.	<ul style="list-style-type: none"> • Solar operated-variable speed direct current (VSDC) system; • Speed of the compressor; • Temperatures and pressures of refrigerating unit.
	<i>Short description:</i> The COP of the VSDC-compressor was 0.38 at the speed of 2500 rpm; the exergy efficiency was 7.4%.	
2016 [18]	VSDC compressor; R134a refrigerant.	<ul style="list-style-type: none"> • AC and DC compressor operated with solar energy; • Inside chamber temperatures; • AC and DC refrigerator energy consumption; • Speed of compressor; • Solar energy generation in peak hours; • Pull down-time.
	<i>Short description:</i> The Solar operated DC refrigerator system installation cost was 18% less than the AC refrigerator; the AC refrigerator consumed high power compared to the DC refrigerator.	
2016 [19]	DC compressor; R134a refrigerant.	<ul style="list-style-type: none"> • Milk cooling system with ice and DC refrigerator operated with solar energy; • Ambient temperature and refrigerator chamber inside temperature; • Battery current and voltage; • Solar irradiance; • PV current and voltage; • Energy consumption of the refrigerator.
	<i>Short description:</i> The refrigerating system was able to produce 12 kg of ice every day.	
2017 [20]	Solar absorption cooling system; NH ₃ -H ₂ O (40:60).	<ul style="list-style-type: none"> • Solar absorption cooling system used for ice storage; • Solar irradiance with time; • PV current and voltage; • Refrigerator chamber inside temperature; • Heat output and flow rate for the cooling system.
	<i>Short description:</i> The COP of the absorption chiller was recorded as 0.43, 0.47 in the March and October months, respectively.	
2017 [21]	12V DC-VCR compressor; R134a refrigerant.	<ul style="list-style-type: none"> • Battery and hybrid mode of load supply; • The current and voltage of solar panel; • Current, voltage consumed by refrigerator; • Solar radiation and battery voltage; • The inside temperature of the DC refrigerator chamber.
	<i>Short description:</i> The refrigerator consumed 15-41% more power in loaded condition than the no-load condition.	
2018 [22]	AC compressor; R134a refrigerant.	<ul style="list-style-type: none"> • Solar operated ice thermal storage air-conditioning and refrigerating system; • PV voltage and current; • Voltage and current of the refrigerator; • The inside temperature of the refrigerating unit; • Battery bank voltage and current.
	<i>Short description:</i> The thermal ice storage was feasible to use instead of the battery bank to store solar energy in the field of distributed photovoltaic refrigeration.	
2018 [23]	12V-24V DC-VCR compressor; R134a refrigerant.	<ul style="list-style-type: none"> • 12V and 24V PV DC refrigerator by varying operating condition; • Compressor run time, current, and power; • The temperature inside the DC refrigerator; • The energy produced by solar panels.
	<i>Short description:</i> The DC-refrigerator work was more efficient at 12 V as compared to 24V.	
2019 [24]	VSDC compressor; R134a refrigerant.	<ul style="list-style-type: none"> • Directly coupled vapor compression (DCVC) solar refrigeration system; • Compressor speed range 1800–4200 rpm; • Current and voltage generated by the PV panel.
	<i>Short description:</i> The COP of the refrigerator was higher (2.25) at lower compressor speed and recorded the lowest (1.85) at higher compressor speed.	
2020 [25]	DC compressor; HC-600a and HFC-134a refrigerant.	<ul style="list-style-type: none"> • Milk cooling with ice and DC refrigerator operated with solar energy; • Milk cooling time and temperature inside the freezer; • Ambient temperature and energy consumption of the refrigerator.
	<i>Short description:</i> The COP of the HC-600a circuit was 1.22; the COP of the HFC134a circuit was 1.24.	

2021 [26]	12V DC compressor; R134a refrigerant.	<ul style="list-style-type: none"> Hybrid (battery + solar) refrigeration system; Vehicle speed and refrigerator energy consumption; Current and voltage of the PV panel.
<i>Short description:</i> The refrigerator energy was lower in the winter season as compared to the summer season at thermostat position 5. The vehicle weight was reduced by 20 kg which enhanced its traveling distance up to 16 km.		

The AC operated refrigerator is consumed 25-40% more power as compared to the DC-refrigerator [17-30]. The refrigerating system's performance depends on the metrological conditions and thermostat position; but, no information about the integral system of the SAREV has been found in the past literature studies. Therefore, the present research work focuses on the integral system performance coupled in the electric vehicle. The integral system of SAREV is built up, and field tests are carried in different conditions.

II. DESCRIPTION OF EXPERIMENTAL SETUP

The experimental set-up consists of a 12 V DC refrigerator (240 L), solar energy production unit (48 V & 600 W), Pulse width modulation (PWM) solar charge controller, and battery bank (48 V & 105 Ah). The refrigerator is fitted on the backside of this vehicle purchased from Phocos India Solar Pvt Ltd. The solar charge controller and PV panels were purchased from Su-Kam Power Systems LTD. The PV panels are installed on the vehicle's roof to receive the maximum amount of solar radiation on sunny days and charge the battery bank in the daytime. These four panels are connected in series. The solar charge controller is 45 A capacity with an auto-recognition system. The vehicle's battery bank used four lead-acid batteries that are connected in series. The battery bank rated at 48 V with the capacity of 105 Ah is used for energy storage and used as a power source for running the compressor and brushless direct-current (BLDC) motor. The battery protection circuit regulates the flow of charge and protects the battery bank from getting overcharged or getting deep discharged. The vehicle's refrigerator is driven by a DC compressor (K35 DC ROHS Sol-cool) operated in 12-12.6 V. A 48-12 V DC converter is used to supply the sufficient low voltage for the refrigerator. The low voltage disconnect (LVD) is used to control the energy fluctuations of the refrigerator. The LVD cuts off the energy supply from the refrigerator when the battery voltage reaches a critical level. A simple mode power supply battery charger (SMPS-15 A) was purchased from Fujiyama Power Systems PVT. LTD that used to charge the battery bank at night time.

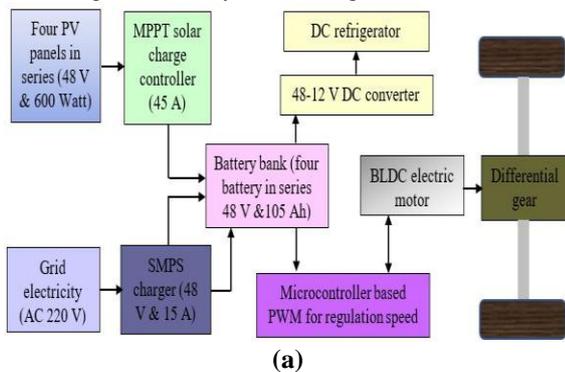


Fig. 1. Example setup (a) block diagram (b) of SAREV.

A solar pyranometer is used to measure the solar radiation falling on solar panels' surfaces. The complete set-up of SAREV is shown in figure 1 (a & b).

A. Experiment Conducted and Test Strategy

To determine the main objectives of this study, a series of experiments were conducted on the vehicle's refrigerator with the hybrid energy mode (battery bank + solar energy). These experiments were completed at the Department of Mechanical Engineering, National Institute of Technology in Jalandhar city of Punjab state in India. These experiments were performed in the summer season (on 2-26th May 2020). All experiments are performed on the refrigerator in the vehicle's stationary condition. The refrigerator experiments are performed in the vehicle's stationary condition. Although the vehicle tests are performed on road condition. The refrigerator's walls temperatures are monitored with six PT100 thermocouples were purchased from Jiamin instrument CO. LTD. These thermocouples are positioned in the upper, middle, and lower parts of refrigerator compartments. The ambient air temperature is measured by one another thermocouple. The refrigerator voltages, currents, power, energy consumption, and all seven temperatures are recorded every 15 minutes time interval. Two digital watt-meters (DC PZEM-051) are used to measure the output energy of solar panels and the DC refrigerator's energy consumption. These watt-meters were purchased from Unitrend technology CO., LTD. The refrigerator experiments were divided into three steps. At the first step, the performance of PV panels was tested. The vehicle's refrigerator was tested by varying the thermostat position at the no-load condition in the second step. In the third step, the refrigerator's performance was tested in different load conditions (15 L & then 25 L). The water is used as a test material in the refrigerator loading experiments. The following experimental conditions were maintained across this study:

- The refrigerator chamber door was kept open for 8 hours to maintain thermal equilibrium with ambient air before starting each experiment.



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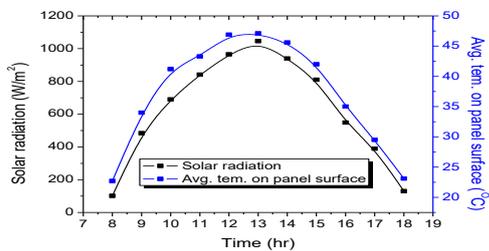
- The door of the refrigerator was kept closed during the experiment time.
- The battery bank of the vehicle must be fully charged before starting the experiment.
- The solar panels were always kept dust-free and no shade on them throughout the day.
- All these experiments were performed on the same location and same road route in sunny days conditions.

III. RESULT AND DISCUSSION

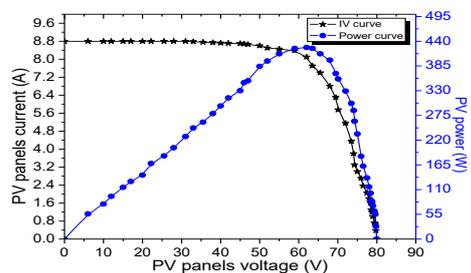
The temperature variation on the solar panel surface is shown in figure 2(a). The test period of the PV panel was selected from 8:00 a.m. to 6:00 p.m., due to the maximum solar radiation intensity available in this period.

The intensity of solar radiation finds a lower value (205 W/m^2) in the morning time, and it extends to its maximum value (1050 W/m^2) at noontime.

After that, it starts decreasing with the sunset until it goes to zero. The PV panel surface temperature



(a)



(b)

Fig. 2. The PV panel (a) surface temperature variation (b) IV and power curves.

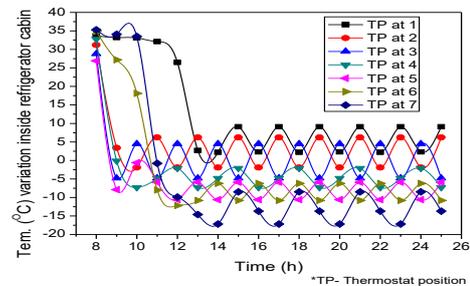
increases with sunrise, and it reaches 45.3°C temperature maximum at noontime and then decreases with sunset time. The temperature recorded on the solar panel surface is higher during the sun's peak hours than sunrise and sunset. The ambient air temperature increases with sunrise and reaches 44.2°C maximum at noon, and decreases with sunset. The IV and power curves of the PV panel on a typical day are shown in figure 2(b). Due to the variation in solar irradiation on a typical sunny day, the PV current, voltage, and power output change in the range of 1.2-8.8 A, 10-62 V, and 55-425 W, respectively. The temperature variation rate is recorded slow on the solar panel surface compared to ambient air temperature variation. The solar energy generated yields about 2.5-4 units (kWh) of energy in a day time.

A. The Refrigerator Performance Test

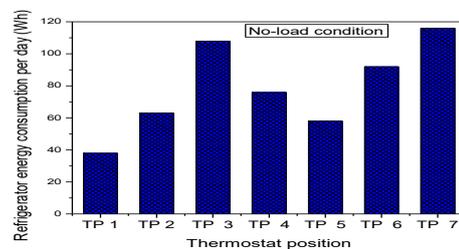
The refrigerator performance experiments are tested at three different conditions: (1) without load, (2) with 15 L

load, and (3) with 25 L load. These experiments are conducted with varying thermostat switch positions from 1-7 that affected the energy consumption of the refrigerator. The thermostat positions have been labelled on positions 1-7 to maintain the desired cut-off temperature from $+6$ to -16°C inside the refrigerator chamber. The refrigerator's circuit stops the power supply from running the compressor at a thermostat position zero. The performance of refrigerator parameters is studied through the different plots that are elaborated in the next section.

- The Performance of Refrigerator at No-load Condition: The refrigerator chamber is kept empty for conducting no-load experiments. figure 3(a) shows the average temperature variation profile for the different thermostat positions (1-7) inside the refrigerator chamber. The refrigerator compressor is permitted to run nonstop for one day on each thermostat position in the experimental period. The refrigerator compressor was allowed to run nonstop for one day on each thermostat position in the testing period. At thermostat position one, the refrigerator's inside temperature reaches 2.1°C in 5.1 hrs. The refrigerator's compressor is switched off after reached 2.1°C and automatically started running at 9°C . The cycling process occurs due to automatic the on/off of the compressor. For the thermostat positions 2, 3, 4, 5, 6, and 7, it takes 1 hr, 1.5 hrs, 2 hrs, 2.1 hrs, 4.3 hrs, and 6.5 hrs respectively for the initial pull-down cycle. The lowest temperatures are recorded as -1.9°C , -6.2°C , -7.3°C , -10.6°C , -12.4°C , and -17.7°C for the thermostat positions from 2 to 7 respectively. Refrigerator power consumption is illustrated in figure 4(b). Equations 1 & 2 are used to calculate the power and energy consumption of the refrigerator.



(a)



(b)

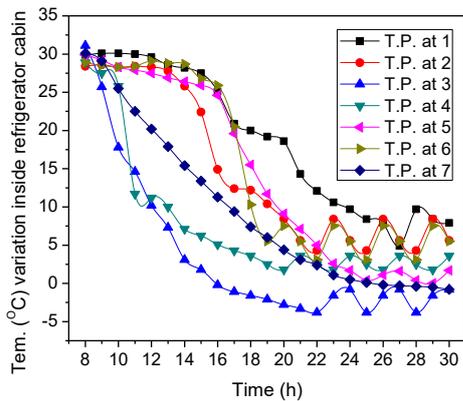
Fig. 3. The refrigerator (a) inside temperature variation (b) energy consumption.

$$Power = Current \times Voltage \tag{1}$$

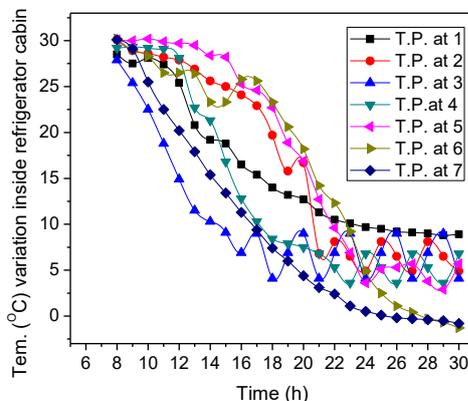
$$E_{daily} = \int_0^{24hr} P \cdot dt \sum_{t=0}^{24hr} p \cdot \Delta t \tag{2}$$

P is the power consumption of the DC compressor, and dt is the test run time. The refrigerator energy consumption is recorded as 38 Wh, 63 Wh, 108 Wh, 76 Wh, 58 Wh, 92 Wh, and 116 Wh for the thermostat positions from 1 to 7. The refrigerator's inside temperature was decreasing by increasing the thermostat position from 1 to 7. The refrigerator was consuming higher energy at the 7th thermostat position. At the 5th thermostat position, the refrigerator maintains the lower desired temperature within the minimum time-period by consuming less energy.

▪The Refrigerator Experiment with 15 L Load Condition: Figure 4(a) shows the average temperature variation inside the refrigerator chamber with the 15 L load at different thermostat positions. The refrigerator chamber was filled with 15 liters of water for conducting experiments. The refrigerator door closed after measuring the initial temperature of the water. The refrigerator's compressor was run continuously during experiments at different thermostat positions from one to seven. The inside temperature of the refrigerator was decreasing at a faster rate on the 3rd



(a)



(b)

Fig. 4. The refrigerator experiment with (a) 15 L (b) 25 L load conditions.

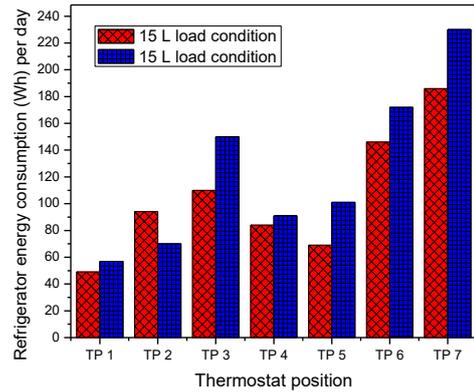


Fig.4(c). The refrigerator energy consumption for 15 L and 25 L load conditions.

thermostat position. The refrigerator's initial pull-down cycle time was smaller at the third thermostat position than the other thermostat positions.

▪The Refrigerator Experiment with 25 L Load Condition: Figure 4(b) shows the average temperature variation inside the refrigerator cabin with the 25 L load at different thermostat positions. The refrigerator chamber was filled with 25 liters of water for conducting the experiments. The refrigerator's compressor was run continuously during experiment time at different thermostat positions from 1-7. The cabin's inside temperature decreased faster on the 3rd thermostat position than the other thermostat positions. The 25 L load cycle started at a higher temperature than the 15 L load condition. However, the energy consumption was higher in 25 L load experiments. The cooling rate was recorded lower in the 25 L load condition than the 15 L load condition in the same time interval.

▪The Refrigerator Energy Consumption for 15 L and 25 L Load conditions: Figure 4(c) shows the energy consumption of the refrigerator with load variation at different thermostat positions. The energy consumption of the refrigerator was depending on the compressor running time and thermostat position.

The refrigerator was consuming higher energy at thermostat position seven in all load conditions. The 15 L load condition's energy consumption was recorded as 49 Wh, 94 Wh, 110 Wh, 84 Wh, 69 Wh, 146 Wh, and 186 Wh for 1-7 thermostat positions. However, the 25 L load condition's energy consumption was recorded as 57 Wh, 70 Wh, 150 Wh, 91 Wh, 101 Wh, 172 Wh, and 230 Wh for 1 to 7 thermostat positions, respectively. The minimum energy consumption was recorded at the fourth and fifth thermostat positions.

▪The Refrigerator Power Consumption at the 5th Thermostat Position: The refrigerator power consumption with load variation at thermostat position five is shown in figure 5. The power consumption was increasing with

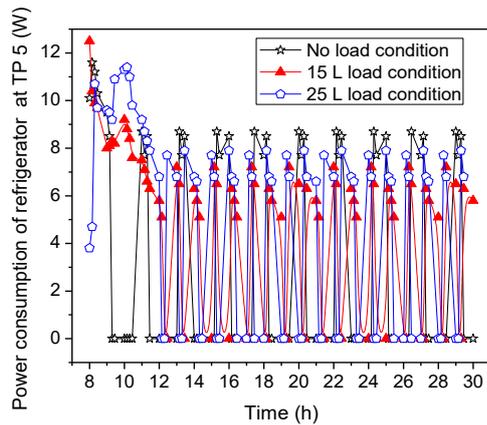


Fig. 5. The refrigerator power consumption at 5th thermostat position.

increasing the load condition. The higher power consumption was 11.6 W recorded for each load condition at thermostat position 5. The initial power consumption was higher in each load condition; after that cycling process occurred (compressor on/off starts).

▪ **The Refrigerator COP Variation in No-load condition:** The refrigerator COP variation with compressor speed is shown in figure 6. The refrigerator COP was decreasing with the increasing compressor speed. The COP was recorded higher (1.32) at the lower speed (2000 rpm) and lower (1.17) at the higher speed of the compressor (3600 rpm).

▪ **Vehicle Distance travelled with load variation:** The distance covered by the vehicle with load variation in different energy modes is shown in Figure 7. The vehicle was covering a higher distance in hybrid energy mode for each load condition. The travelling distance was decreasing with increasing load on the vehicle. However, vehicle energy consumption was increasing with the increasing load on the vehicle. The vehicle was travelling 68.3 km, 65.3.6 km, 63.4 km distance in no-load, 100 kg, and 200 kg load conditions respectively by consuming 3010 Wh, 3230 Wh, and 3450 Wh energy.

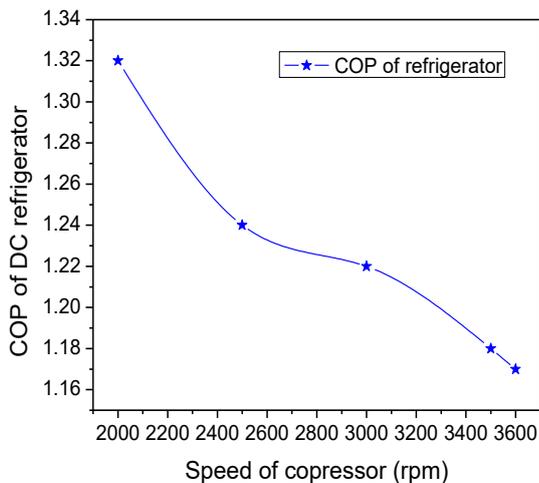


Fig. 6. The COP of refrigerator variation with compressor speed.

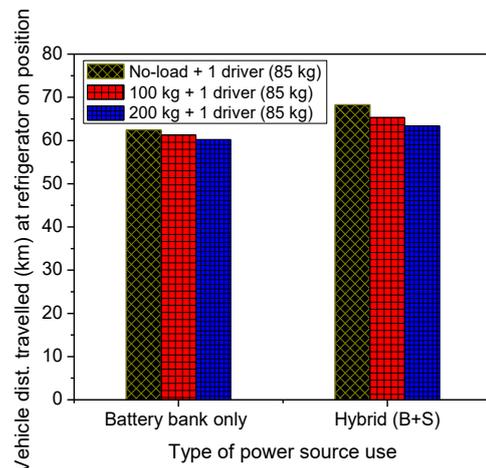


Fig. 7. Vehicle distance travelled with load variation.

IV. CONCLUSION

The solar-assisted refrigerator of the electric vehicle has been presented in this study. The SAREV was designed primarily for street vendors that are facing the problem of perishable food product delivery. The integral system was tested with different modes of energy at different load conditions. The integral system of this vehicle is an economical and energy-efficient technique for intercity cold chain transportation. The essential outcomes of this research work are summarized as follows:

- The solar system of the vehicle was generating 2.5-4 kWh energy on the average sunny day.
- The maximum output current and power of PV panels were recorded as 8.8 A and 425 W, respectively.
- The refrigerator's energy consumption at no-load was varying 38-116 Wh for 1-7 thermostat positions.
- For 15 L load, the energy consumption was varying 49-186 Wh for 1-7 thermostat positions.
- For 25 L load, the energy consumption was varying 57-230 Wh for 1-7 thermostat positions.
- At the third thermostat position, the minimum temperature inside the refrigerator's cabin was recorded as -17 °C, -3 °C, 2.5 °C for the no-load, 15 L load, and 25 L load conditions, respectively.
- The COP of the refrigerator was decreasing with the increasing compressor speed.
- The COP was recorded higher (1.32) at the lower speed (2000 rpm) and lower (1.17) at the higher speed of the compressor (3600 rpm).
- The refrigerator's inside temperature was decreasing faster at the third thermostat position among all load conditions.
- The refrigerator was consuming higher energy at the 7th thermostat position among all load conditions.
- At the higher thermostat position, the battery bank was able to run the refrigerator 240 hr, 96 hr, 72 hr for the no-load, 15 L load, and 25 L load conditions.

- The vehicle was travelling 68.3 km, 65.3.6 km, 63.4 km distance in no-load, 100 kg, and 200 kg load conditions respectively by consuming 3010 Wh, 3230 Wh, and 3450 Wh energy.
- The travelling cost of the vehicle was 1-1.5 INR per kilometer.

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AUTHORS PROFILE



Dr. R. S. Bharj, currently working as Associate Professor, Dr B. R. Ambedkar National Institute of Technology Jalandhar (Punjab), India. He carries academic and administrative experience of more than 25 years. He has supervised 6 Ph.D. and more than 25 M.Tech. students working in the field of Mechanical Engineering. He has authored more than 60 published technical papers, and co-authored book chapters. His areas of interest are Renewable and Appropriate Energy Systems, Alternate Fuels, IC Engines and Automobile Engineering and Electric Vehicles.



Surender Kumar is currently pursuing Ph.D. in the Department of Mechanical Engineering, Dr B. R. Ambedkar National Institute of Technology Jalandhar, Jalandhar (Punjab), India.